## A PRELIMINARY FIRST WALL DESIGN PROCEDURE AND RELEVANT HEAT FLUX ESTIMATION

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#### **OUTLINE**

#### Introduction

Heat load modeling for the engineering systems code Bluemira [8,9]

- Charged Particle model
- Radiation source model

Wall load specification and wall shape design

**Referral project** 

#### **FIRST WALL HEAT LOADS**

**Engineering constraints** 

- Large electricity production
- Heat loads on the first wall
- Superconducting magnets
- Maximum allowable stress on the support structure

Plasma-wall interaction challenge

- No perfect core plasma confinement
- Power crossing separatrix
- Flux lines intersecting wall
- Power loads restricted to a small wetted area
- If all SOL power strikes the divertor plates, peak loads might be intolerable



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HEAT LOAD MODELING FOR THE ENGINEERING SYSTEMS CODE BLUEMIRA

**Goal:** defining a procedure to design a preliminary first wall profile and

estimate expected heat loads as support of the *engineering phase* 

Main requirement: quick and sufficiently reliable

**Current state:** 

- First wall shape design
- First wall heat flux calculation due to charged particles
- First wall heat flux calculation due to radiation





**Optimise First** 

Wall Profile

if FW is not in INPUT:

Make preliminary First Wall profile (addtional geometrical inputs)

Main chamber shapingDivertor parametrization



2-D Heat Flux distribution on the preliminary profile

Double exponential decay [10]

$$q_{p,u}(r_u) = \frac{P_{SOL,n}e^{\left(-r_u/\lambda_n\right)}}{2\pi R(r_u)\lambda_n} + \frac{P_{SOL,f}e^{\left(-r_u/\lambda_f\right)}}{2\pi R(r_u)\lambda_f}$$

Flux expantion

$$f_{x,t} = \frac{dr_t}{dr_u} = \frac{R_u B_{p,u}}{R_t B_{p,t}}$$

Angle between flux surface and first wall

 $q_{\perp,t} = q_{p,t} sin \beta_t$ 



Optimise First Wall Profile

Heat Flux limit (user input)
Local first wall reshaping



#### Outcome:

- Results benchmarked against SMARDDA<sup>[11]</sup>
- t < 1'
- Flexible set of input params
- The module has been widely appreciated and used outside bluemira
- Suitable for a sensitivity analysis/robust design
- Suitable for cross functional team work







Potential steady-state scenario:

- Fusion power ~ 2GW
- Alpha + auxiliary heating to be exhausted ~ 500 MW
  - $\circ$  Core ~ 350MW
  - $\circ ~~\text{SoL} \sim 150 \text{MW}$
- > Detailed calculation of radiation distribution requires high-fidelity modelling
  - \* Time
  - \* Uncertainty need of full understanding on transport processes

Simple assessment:

- Core radiation source
- SoL radiation source
- Coupling with CHERAB
   First wall heat flux

- Core radiation source
- SoL radiation source

$$\frac{dP_{line,Imp}}{dV} = f_{Imp} n_e^2 P_{line-LF,Imp}(n_e,T_e) \longrightarrow$$

T, n Impurity data Atomic database<sub>[6]</sub>

Coupling with CHERAB

#### Core:

- Te > 300eV
  - → low-Z impurities fully stripped → main contribution from bremsstrahlung
- Synchrotron radiation currently not included
  - $\longrightarrow$  less Xe in the core to radiate same amount of power
  - → Radiation distribution may slightly change → peak in the core
  - → No significant impact on the first wall heat flux





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#### Core radiation source

- SoL radiation source
- Coupling with CHERAB
- Two-Point model  $2n_{t}T_{t} = n_{u}T_{u}$   $T_{u}^{7/2} = T_{t}^{7/2} + \frac{7}{2}\frac{q_{\parallel}L}{\kappa_{0e}}$   $q_{\parallel} = \gamma n_{t}kT_{t}c_{st}$   $T_{t} = F_{u}\left(1 + \frac{\varepsilon/\gamma}{T_{t}}\right)^{-2}$
- Exponential decay  $n(r) = n_{\text{LCFS}} \exp(-r/\lambda_n)$  $T_e(r) = T_{e,\text{LCFS}} \exp(-r/\lambda_{T_e})$
- \* [Pitcher, PPCF (1997)]
- \* [Stangeby, IPP (2000)]





#### Outcome:

- t source ~ 3'
- t tracing ~ 30'
- The module has been widely appreciated and used outside bluemira

→ Going to benchmark against MAST-U experimental results



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#### WALL LOAD SPECIFICATION AND WALL SHAPE DESIGN

- From 2D to 3D analysis
- Providing inputs for PFC design
- Found out about H.E.A.T. [6]
- Made contact with Tom Looby
- Promoting the use



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\* Vaccaro, D., Elaian, H., Reimerdes, H., et al., Thermal, electromagnetic and structural analysis of gas baffles for the TCV divertor upgrade, Fusion Engineering and Design 23

## Thank you!

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## **EXTRA SLIDES**

#### Goal:

 Avoiding neutrals generated by recycling at the divertor targets to escape into the main chamber
 Outer poloidal field coils (for plasma positioning and shaping)

**Design consideration:** 

- Thermal loads
- Electromagnetic loads

**Design solution:** 

- Polycrystalline graphite (SGL R6650)
  - $\,\circ\,\,$  32 tiles mounted on the HFS
  - 64 tiles mounted on the LFS



#### Guidelines

- Avoid exposure of leading edges  $\rightarrow$  set-back
- Holes for diagnostics impose most severe constraints  $\rightarrow$  ad hoc baffle
- Maintain compatibility with both magnetic field helicities → symmetric design



#### Shielded effect is provided by two shadow zone

- Upper part of the baffle surface is in the shadow of the wall
- Laterally baffle surface is in the shadow of the adjacent baffle

0.8 R (m)





## Fraction of the plasma current is conducted through the baffle Typical values expected in TCV

*I*<sub>h,max</sub>=250kA

#### Input

- Magnetic field: 1.43 T
- Current: 4kA

#### Output

Volumetric force

TOTAL FORCE  $F_x = 425 N$   $F_y = -0.5 N$  $F_z(vertical) = 1090 N$ 



# The geometry at the screw is optimised to limit baffle deformation, displacement and stresses (mainly in the graphite) to tolerable values

- Stresses lower than graphite limits
- Deformations and displacements between baffles to be avoided
- Movement in vertical and toroidal directions limited by 4mm and 2mm spacing, respectively



# The force calculated by the ANSYS Maxwell Module generates a deformation that is highest at the baffle tip





Maximum tensile stress32 MPaMaximum compressive stress80 MPa

- Both 2x lower than limit

TOTAL DEFORMATION  $D_x = 0.09 mm$   $D_y = 0.04 mm \ll 2mm$  $D_z = 0.25 mm \ll 4mm$ 

Extra slides

Graphite and titanium coefficient of thermal expansion are characterized by close magnitude
 The thermal analysis which simulates baking operations shows that stresses due to thermal expansion do not give cause for concern





• BAKING SIMULATION  $\circ T_{Start} = 22^{\circ}$ 

$$\circ$$
  $T_{Finish} = 250$ 

- Thermal Expansion
  - Graphite: 3.5 x10<sup>-6</sup> C<sup>-1</sup>
  - Titanium: 9.2 x10<sup>-6</sup> C<sup>-1</sup>

PRELOAD AND THERMAL CONDITIONMaximum obtained tensile stressMaximum obtained compressive stress10.76 MPa67.85 MPa
$$SF_{Tensile Stress} = \frac{65}{10.8} = 6$$
 $SF_{Compressive Stress} = \frac{150}{67.9} = 2.2$ 

#### **HF CALCULATION – BENCHMARK DETAILS**

Test on DEMO baseline 2017 and benchmark against SMARDDA(4) (Psol = 69MW, Pinner = 50%, Pouter = 50%)



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