

NSTX-CALC-11-05

Thermal Analysis of NSTX Center Stack and Outboard Divertor Tiles

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NSTX

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BOARD DIVERTOR TILES**Keywords: NSTX Plasma Facing Components, Graphite Tiles, Thermal Analysis

Abstract

Thermal analysis was performed on the Center Stack[CS] and Divertor tiles assuming that the CS tiles would be thermally isolated from the stack and the heat transfer would be by radiation only, while the divertor would be cooled by conduction into the stainless steel mounting plates.

A one dimensional, transient heat transfer program [TILE.TEMP.1] was written in QUICKBASIC to derive the temperature vs time, in four layers of graphite tiles in the NSTX center stack. The temperatures in both the CS and the divertor were analyzed by P-THERMAL 2.5.[1]

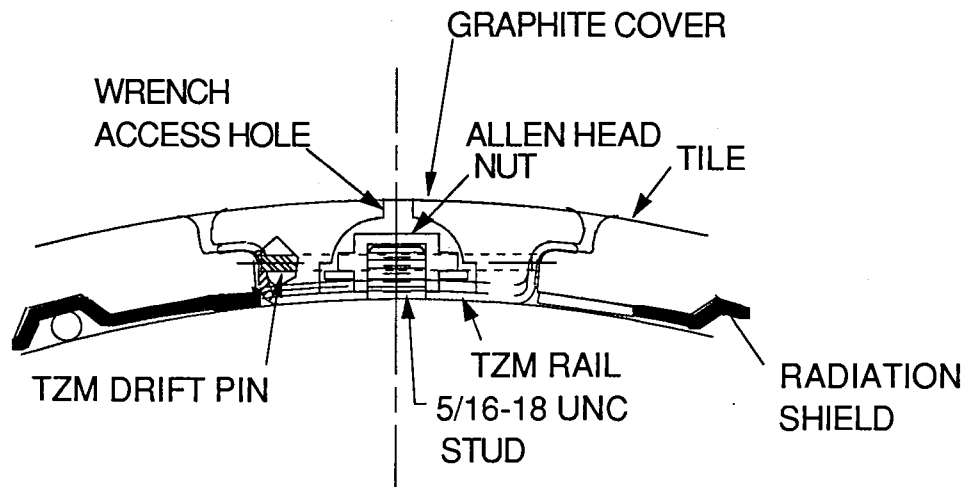
The BASIC program was designed to give a rough approximation of the thermal profile of a 1-D tile. It allowed quick turnaround and served as a guideline during conceptual design of the tiles and, later, as a litmus test to validate thermal analyses done with commercial finite element (FEA) codes. The BASIC code was in fairly good agreement with the commercial codes and peak temperatures calculated for the two varied by only 10%.

Center Stack Thermal Analysis**Overview**

The BASIC code was derived using the solution for transient heat transfer in a 1-D, semi-infinite plate, exposed to a constant flux, as the starting point. This solution is relatively accurate for 1.25 cm thick tiles for the first 2-3 seconds of a five second pulse,. This corresponds to the time period during which the temperature effects are small and the rear of the plate is relatively unchanged. Time periods exceeding this result in increasing inaccuracies.

The solution for an infinite plate assumes that the material properties, i.e. specific heat, thermal conductivity, and diffusivity remain unchanged with temperature. Of course, these constants change very dramatically in graphite materials so temperature compensation was required. An iterative approach was used, calculating the mean temperatures and corresponding material

FIGURE 1. BASELINE DESIGN



FEATURES

- RAIL BOTTOMS OUT ON EMBOSSED FEET
- RAIL RETAINS & INDEXES TILES
- TILES COMPRESS WAVE WASHERS

material properties over that time period. A second compensation method was also employed whereby the energy balance was checked and the temperature profile was shifted by an exponential function until balance matched within 10%. The layers were weighted by an algorithm according to distance back and time into the pulse (penetration into the tile). This procedure does not give exact solutions for the boundary conditions, but it is less time consuming, much less complex than a true numeric solution, and it is more accurate than simply assuming an overall average bulk material property. As will be seen, it gives adequate results for first cut designs.

Description

Analysis was done assuming temperature dependent material properties of a known graphite and using tile thickness, flux, heating time, cooling time, center stack temperature and torus temperature as input variables. The early runs were done assuming POCO, FMI-4D, and ATJ graphites. Later, analyses were added to include Allied Signal’s composite graphite product 865-19-4 C-C (AS) when its advantages became evident. Output of the code was temperature of each layer at each iterative time period, mean value material properties of the layers, total heat in, calculated heat balance, radiative heat to center column, and a graph of temperature vs time for each layer. The specific heat, thermal conductivity, and diffusivity were graphed and then curve fit with higher order polynomials, which were used by the code to solve for the material constants at each time iteration (typically 0.1 seconds). The same procedure was used to obtain the Gaussian error function for arguments from 0 to 2. The code was run until the peak temperature and cool down temperature reached steady state. During the cool down period the tile was assumed to have a constant(bulk) temperature through its thickness. This can be shown to be an acceptable assumption for slow radiative heat loss using the criteria for "lumped analysis".[2]

Tiles were assumed to have no thermal contact with the center stack, that is, all loses were radiative. One layer of heat shielding was assumed on the center stack side of the tile. During cool down the material properties were recalculated for each time iteration (typically 4 seconds)

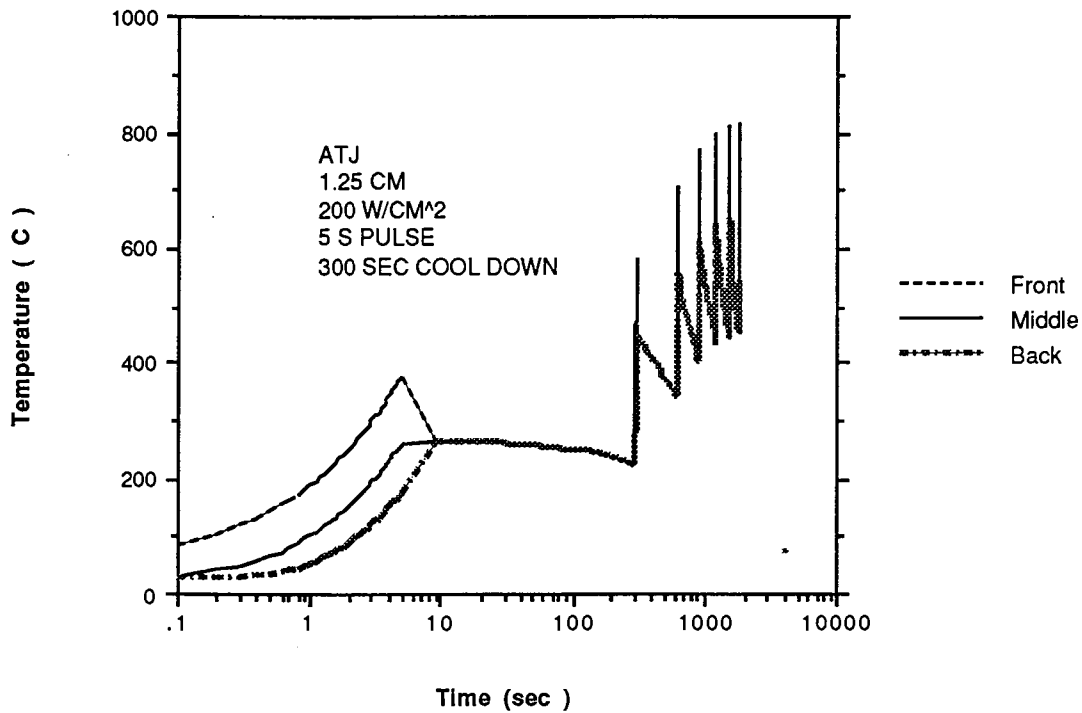
A typical run for a 1.25 cm tile is shown in Fig. 2.

* FMI-4D refers to the coarse weave carbon-carbon graphite manufactured by Fiber Materials Inc.

Typical Runs

Material	<u>POCO</u>	<u>POCO</u>	<u>POCO</u>	<u>FMI-4D*</u>	<u>ATJ</u>
Thickness	1.25	2.54	5.08	1.25	1.25
Heat flux (w/cm ²)	320	320	320	320	320
Pulse length (sec)	5	5	5	5	5
Cooling Time (sec)	300	300	300	300	300
Center stack and torus temp. (K)	300	300	300	300	300

Figure 2. Tile Temperature as a Function of Repetitive Cycling



Results- steady state values

Material	<u>POCO</u>	<u>POCO</u>	<u>POCO</u>	<u>FMI-4D</u>	<u>ATJ</u>
Thickness	1.25	2.54	5.08	1.25	1.25
Peak face temp (C)	1210	1283	1330	1124	1176
Peak rear temp during pulse (C)	638	636	685	682	657
Peak bulk temp. after shot (C)	865	818	794	865	861
Bulk temperature at cool down C)	523	624	684	540	514
Time to steady state (cycles)	6	8	12	6	6
Peak radiative heat to center column	0.5 w/cm ²				

A run performed with a 1.25 cm tile (POCO) and the torus and center stack temperature held at 473 K results in the following:

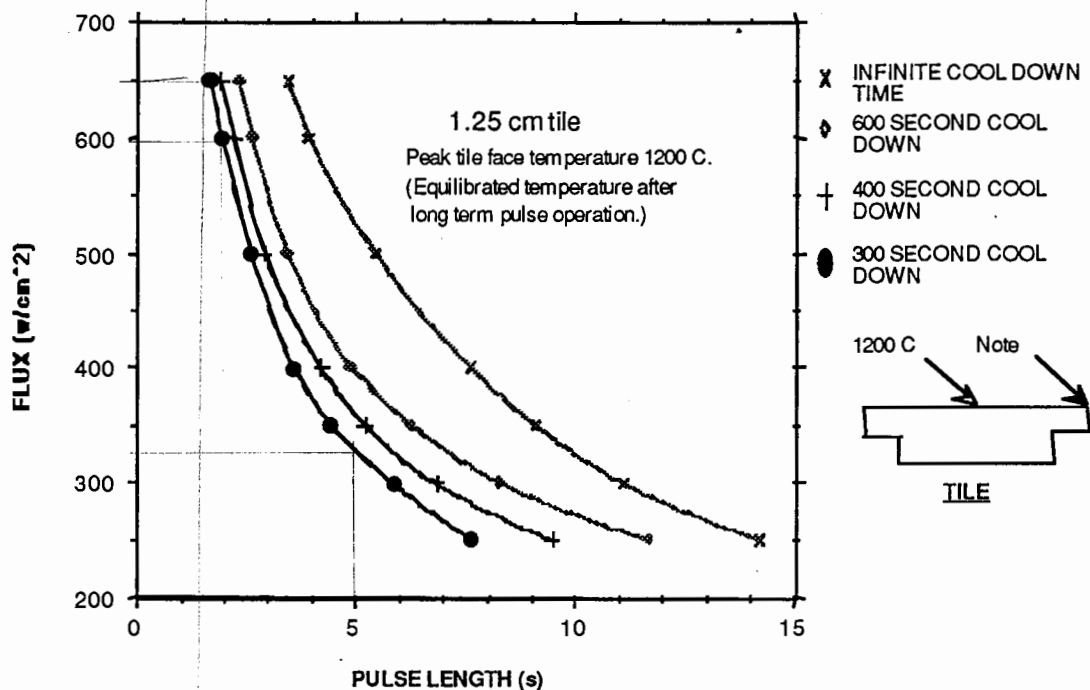
Peak face temperature 1220
Peak rear temperature 693

Peak bulk temp. after shot 901
 Bulk temp. at cool down 544
 Time to steady state 6 cycles

As might be expected here is very little change in the peak parameters when the environment is operated at 200 C instead of 27 C, this being due to the fact that radiation is considered as the only heat loss and there is little difference in the fourth powers of the temperature differences.

A comparison was done using TILE.Temp.1 to relate pulse times, flux levels and cool down periods which result in a given face temperature of 1200 C. The result is an operating envelope for the NSTX device, assuming that the limiting factor is the maximum temperature of the center stack tiles. Results are plotted in Fig.3.

Fig. 3 PULSE LENGTH AS A FUNCTION OF FLUX AND COOL DOWN PERIOD

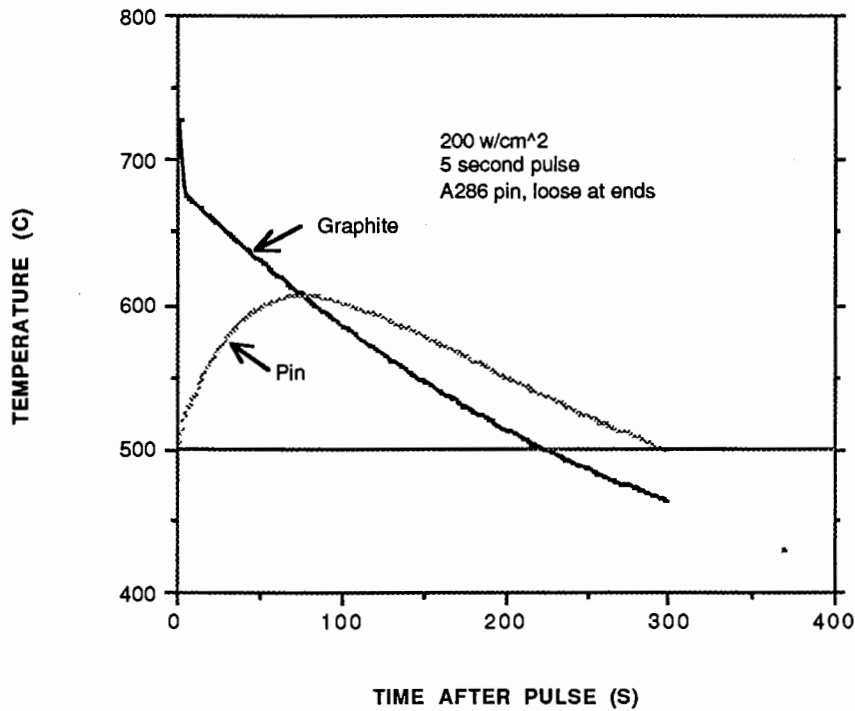


Note: Temperature at corner will run approximately 17-20% higher than face.

Drift pin temperature response

The center stack basket drift pin temperature response was analyzed at steady state with a 200 w/cm² heat load and 5 minute cooling periods. To be conservative the pin was assumed to be isolated from the bracket. The results are shown in Figure 4.

FIGURE 4. PIN TEMPERATURE RESPONSE AFTER PULSE DURING STEADY STATE OPERATION



Comparison to other tile thermal results

A run was performed using the parameters of the PPPL Thermal Analysis Document:[3]

MATERIAL	ATJ
Thickness	1.3 cm
Constant heat flux	160 w/cm ²
Pulse length	5 sec
Cooling Time	300 sec
Center stack and torus temperature	323 K

Cycling was continued until the tile peak temperature and the bulk temperatures between shots reached equilibrium.

<u>Results</u>	<u>PPPL</u>	<u>TILE.TEMP.1</u>
Peak face temperature	837 C	770
Peak rear temperature	-	546
Bulk temperature	520 C	445

P-THERMAL Runs

- Several 2-D runs were done on P-THERMAL, after the initial design phase was complete, to verify the thermal performance of the tiles.
- 3-D runs of the actual tile model [ProE][4] were also performed. These analyses took into account the detail topographical features such as holes, notches, and asymmetry.
- An additional set of 3-D runs were made of the tile models to determine the effects of changing material to AS graphite and reducing the heat flux.

Summary of Typical Runs - steady state results

Material	<u>POCO</u>	<u>POCO</u>	<u>FMI-4D</u>	<u>AS</u>
Model	2-D	2-D	3-D	3-D
Thickness (cm)	1.15	5.08	1.15	1.15
Constant heat flux (w/cm ²)	320	320	320	200
Pulse length (s)	5	5	5	5
Cooling Time (s)	300	300	300	300
Center stack and torus temperature (K)	373	373	373	423

Results- steady state values

Material	<u>POCO</u>	<u>POCO</u>	<u>FMI-4D</u>	<u>AS</u>
Model	2-D	2-D	3-D	3-D
Peak <u>face</u> temp during pulse (C)	1340 C	1335	1160	906
Peak corner temp during pulse(C)	1670 C	1383	1160	982
Peak temperature at holes(C)	-	-	1262	906
Bulk temp at cool down(C)	540	540	540	530

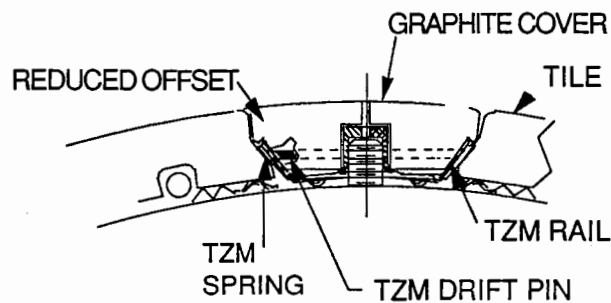
Discussion

There seemed to be no advantage in thicker tiles from a thermal consideration, in fact the equilibrated face temperatures and the bulk temperature between shots actually rose, due primarily to the inability of the large mass to radiate away heat between shots. There is some small advantage in the thicker tiles in that they do take a longer time to reach equilibrium (12 cycles vs 6) and the device could be operated slightly longer before maximum temperature would be reached. After this point, however, a lengthy shutdown would be necessary to allow removal of the heat. The results indicated that the 1.25 cm rail tiles, at a heat flux of 320 cm² were close to the design limit of 1200 C at the face but there was a considerable gradient [330 C] to the edges of the tiles

with overhang. See Fig.3. This was unacceptable and various design changes were studied to find a solution for this effect. Among these were:

- Eliminating the overhangs and using a tapered [beveled] interface. This would essentially eliminate the offset and lower the peak temperatures. The problem is that to do this both rail and tile would have to have hard stops against the stack, otherwise there would be no firm indexing point to align the tile faces. This would be possible by retaining the tiles with spring fingers (Fig. 5) which would hold the tile firm against the center stack but permit thermal flexibility. This concept proved to be very difficult and expensive. The fingers had to be separate components, the full width of the tiles, to prevent over stressing, and even then required very tight tolerances, thin gauges and intricate shapes.

FIG. 5 CENTER COLUMN - BEVELED TILE CONCEPT



FEATURES

- SEPARATE SPRING RETAINS TILES
- SPRING MUST BE FULL WIDTH OF RAIL AND THINNER GAUGE
- RIVETS OR SPOT WELDS TO RAIL CHANNEL

CONCLUSIONS

- SPRING THEORETICALLY WORKS
- TIGHT TOLERANCES AND PRECISION DESIGN REQUIRED
- EXPENSIVE, INTRICATE

- Bolting all the tiles just as in the divertor and passive plate designs

This would require doubling the number of center stack studs and required composite graphite for all tiles (machining considerations).

The only practical design solution found was a compromise in which the inner and outer tile overhang radii were increased. This had the effect of shortening the thermal path and also reducing the incidence angle of the plasma flux on the tile edge.

Analysis showed that the result was marginally within the 1200 C constraint at 320 w/cm² and well within if the thermal loads were reduced to 200 w/cm².

Pin temperatures were shown to stay under 600 C during pulse operation and return to a temperature of under 500 C between shots, acceptable temperatures for inconel alloys.

Conclusion

ATJ and POCO are good candidates for the standard tiles but not for rail tiles. Machining considerations, stress concentrations [4],and excessive temperatures at the rail overhang eliminate all but composite graphites for the application. The standard tiles do not have pins or the overhang on the sides and consequently the stresses are lower and temperatures are more nearly constant across the face. A set of analyses were done on P-THERMAL to determine if changing to AS with its higher conductivity, the heat load reduced to 200 w/cm², and the modified radii would alleviate the problem in the rail tile. The results are shown in the attached figures and indicate that everything was well within 1200 C. A decision was made to utilize the Allied Signal material, to gain the advantage of its lower price and shorter procurement time as well as its higher thermal conductivity and subsequent lower temperature gradients. The results showed that TZM was not required for the brackets and pins, and they were changed to inconel 600.

Outboard Divertor Thermal Analysis

Overview

P-THERMAL runs were done on ATJ divertor tiles with thickness of 1 inch (2.54 cm).The flux was applied in an exponential distribution (Fig 6) $q = q_{peak} * e^{-s/\lambda}$ with a λ value of 1.5 to 3 cm. Perfect contact with a Dowtherm cooled backing plate (150 C) was assumed. Runs were continued until the peak temperatures and the cool down temperatures equilibrated.

Runs:

Axisymmetric 1-D and 2-D models
Start temperature 150 C
Pulse length 5 seconds
Cool down periods 300 seconds

Results

1-D analysis indicated peak temperature of ~1200 C for less than 600 w/cm². 2-D analysis showed similar limit, but gave lower temperatures than 2-D due to poloidal conduction along tile.

2-D results:

Thickness cm	Lamda cm	Peak Flux w/cm ²	Peak Temperature C
2.54	1.5	1860	2400
2.54	3	920	1500
2.54	4.5	620	1100

The design point for a double null plasma is a peak flux of approximately 1700 w/cm², which is not possible unless some means such as sweeping is used to reduce the apparent flux. The effect of sweep on apparent flux with Lamda =3 is shown in figure 7 and 8. The results are plotted as peak flux vs power flux width in figures 9 and 10, for an infinite distribution on the divertor and also as a truncated plasma distribution where the width is assumed cut off at the e-folding distance.

Conclusions

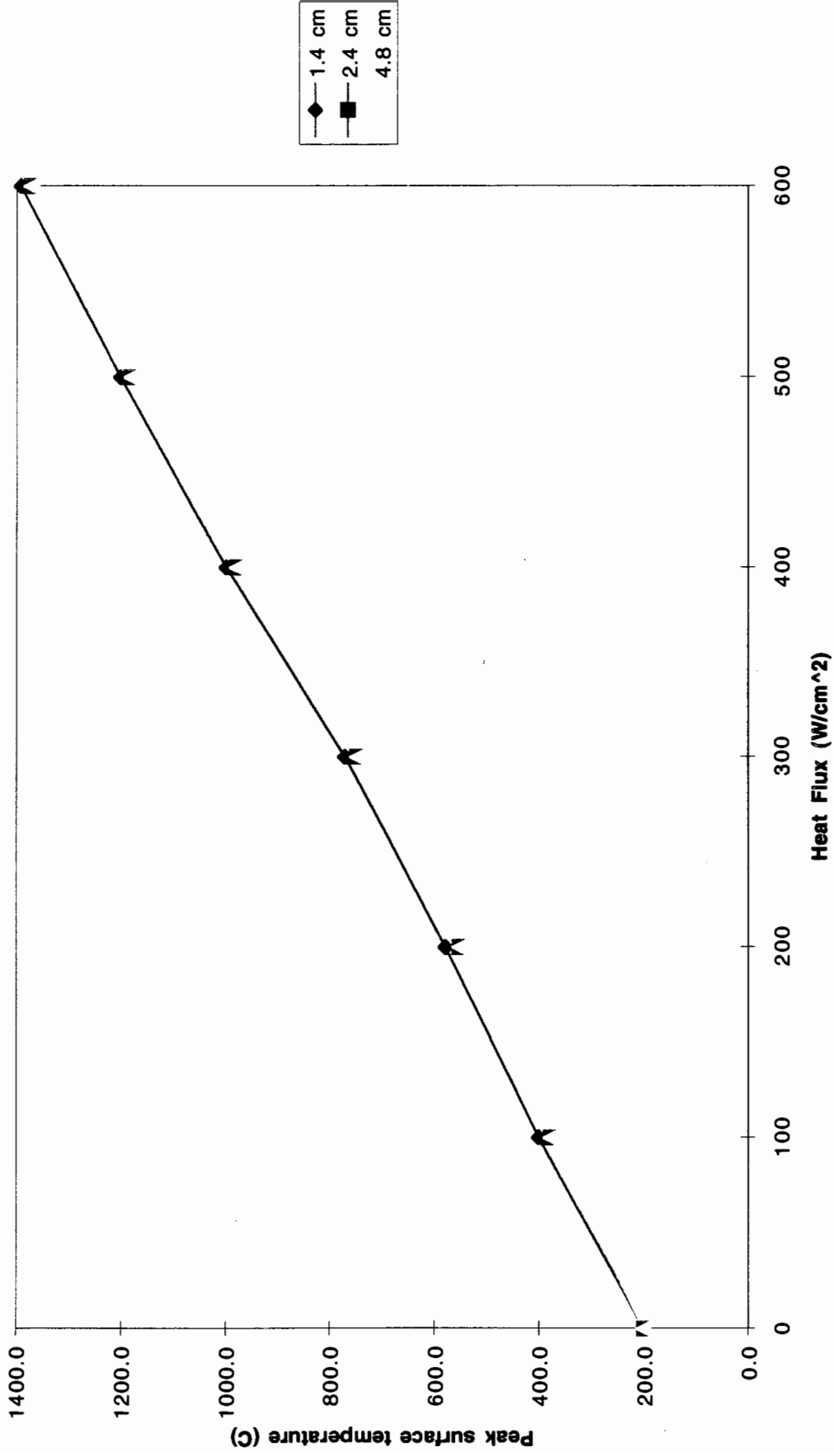
The analyses showed that the divertor tiles could not be operated within a 1200 C maximum temperature envelope unless the apparent peak flux was limited to approximately 600 w/cm². This was shown to be possible for a sweep of 10 cm and $\lambda = 3$ cm. The divertor tiles returned to a bulk temperature of 150 C after the three minute cool down period.

3-D and, 1-D models were run to determine the effect of thicker tiles and the results of both indicated that the major thermal penetration during a 5 second pulses was approximately 0.5 inches, independant of the tile thickness. This indicated that thermal considerations were not a design driver for divertor tile thickness. A one inch tile envelope was adopted, purely out of mechanical attachment considerations.

REFERENCES

- [1] P-THERMAL is a trademark of PDA Engineering Corp.
- [2] D. R. Pitts, *Theory and Problems of Heat Transfer*, Schaum's Outline Series, p. 75, McGraw-Hill Publishing Company, New York, NY, 1977.
- [3] A. Brooks, *Center Stack PFC Thermal Analysis/Fault Analysis*, PPPL Report 13-970217-AWB-01, Princeton, New Jersey, 17 February 1997.
- [4] Pro/Engineer is a trademark of Parametric Technology Corporation.
- [5] Goranson, Nelson, *NSTX Center Stack Tile Analysis and Testing*, LMES Report 11-980501-PLG-02, May 1998.

Peak Surface Temperature vs Heat Flux,
for various tile thicknesses, 1-D calc.
ATJ, 5 seconds on, 300 seconds off,
Fixed equilibrium temperature of 200C

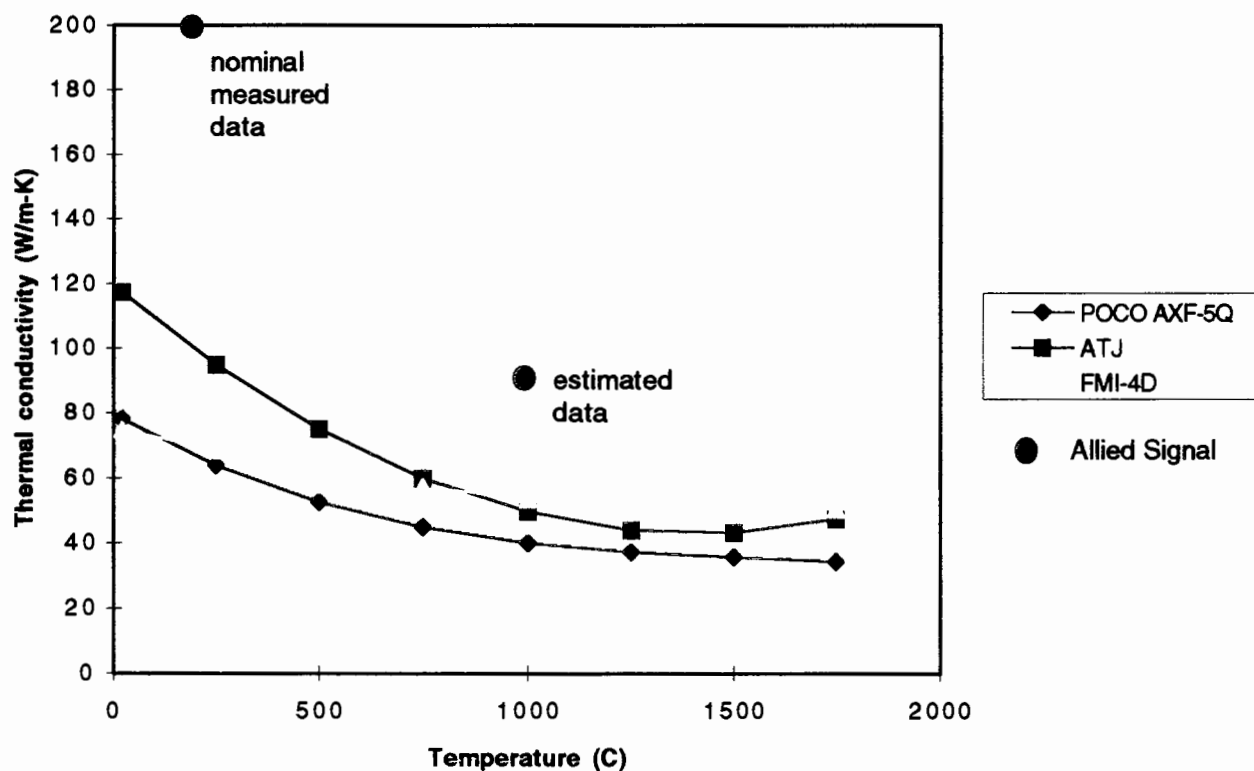


Thermal conductivity comparison for different graphites

Thermal Conductivity, (W/m-K)				
Temperature	POCO AXF-5Q	ATJ	FMI-4D	Allied Signal
20	78	117	75	200
250	64	95	68	
500	52	75	62	
750	45	60	57	
1000	40	50	54	100*
1250	37	44	51	
1500	35	43	50	
1750	34	47	51	

* estimated data

Thermal conductivity of various graphites and CC composites

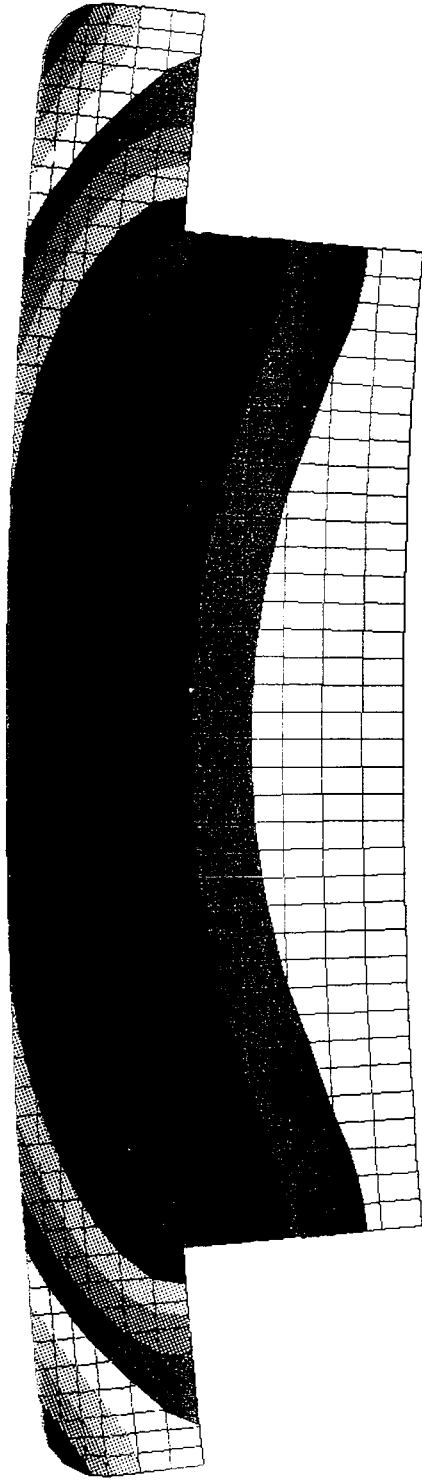


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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**2

1704

1657

1609

1562

1514

1467

1419

1372

1324

1277

1229

1181

1134

1086

1039

Unit = deg C

991

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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**

1898.

1881.

1864.

1847.

1830.

1813.

1796.

1779.

1762.

1745.

1728.

1711.

1694.

1677.

1660.

Unit = deg C

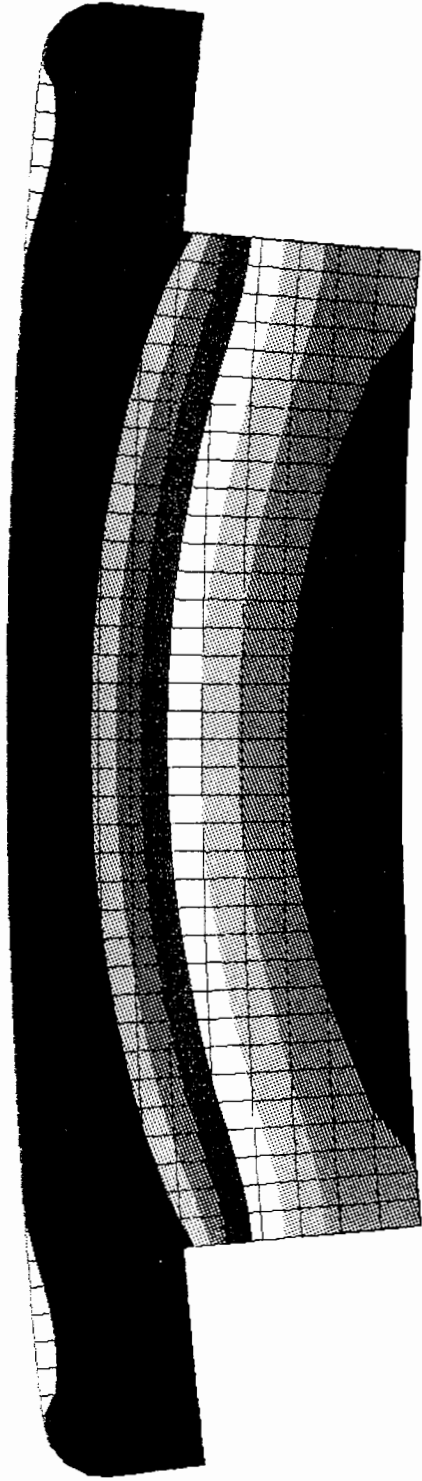
1643.

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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**2

Unit = deg C

941

940

939

938

938

937

936

935

934

933

932

931

930

929

928

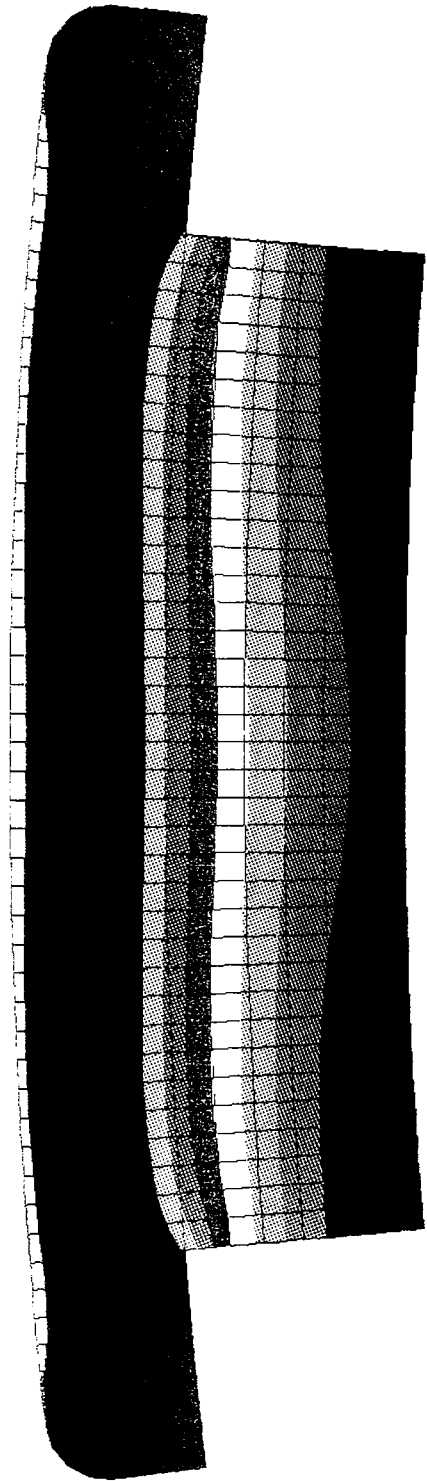
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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**2

1138

1137

1135

1134

1132

1131

1129

1128

1127

1125

1124

1122

1121

1119

1118

Unit = deg C

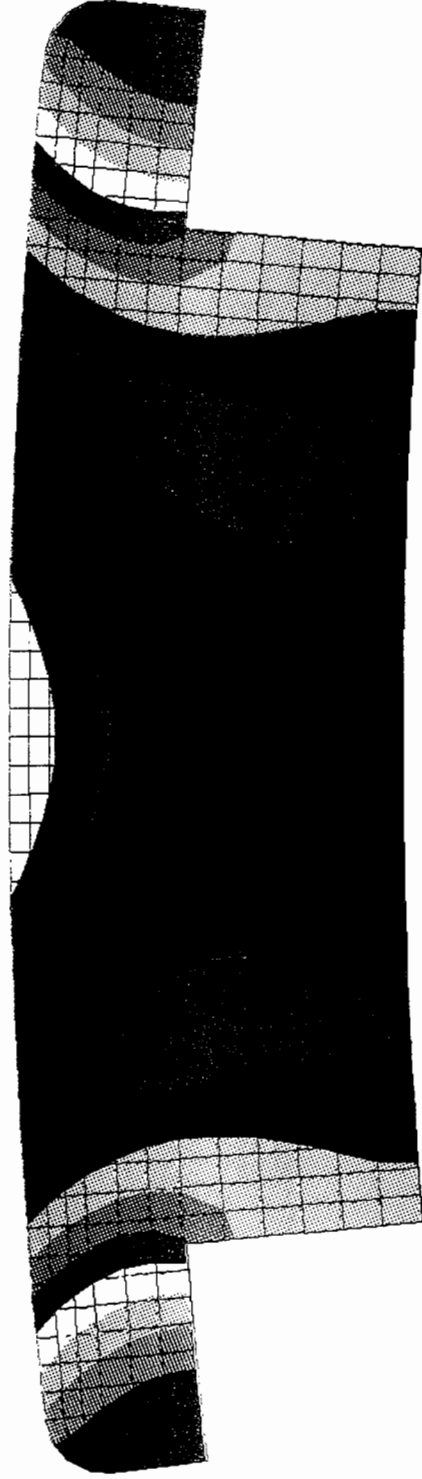
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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**2

1199

1194

1189

1185

1180

1176

1171

1166

1162

1157

1152

1148

1143

1139

1134

Unit = deg C

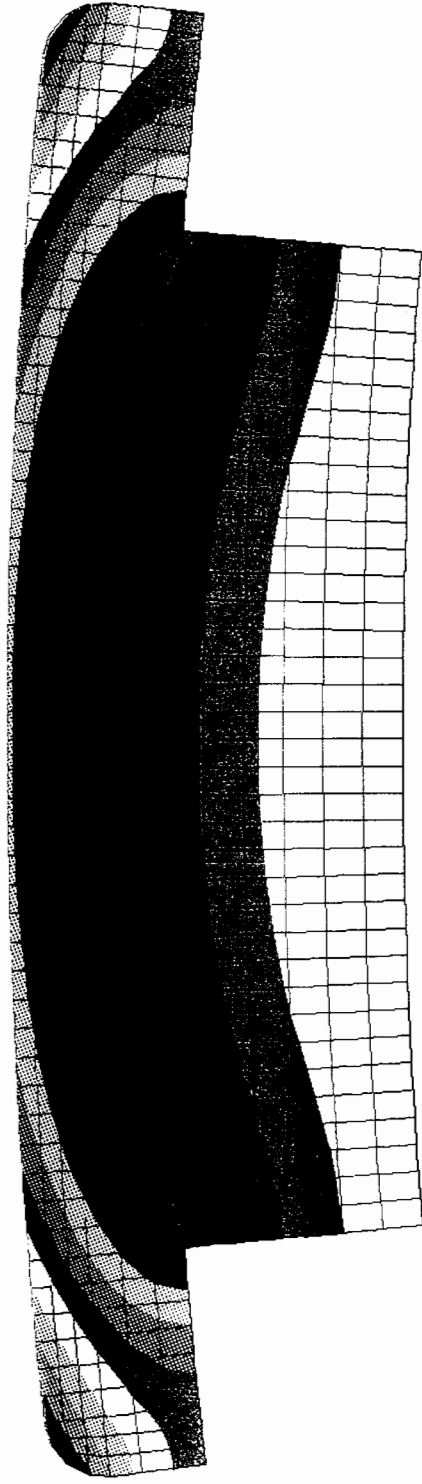
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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**2

Unit = deg C

1396

1356

1316

1276

1236

1196

1156

1116

1076

1036

996

956

916

876

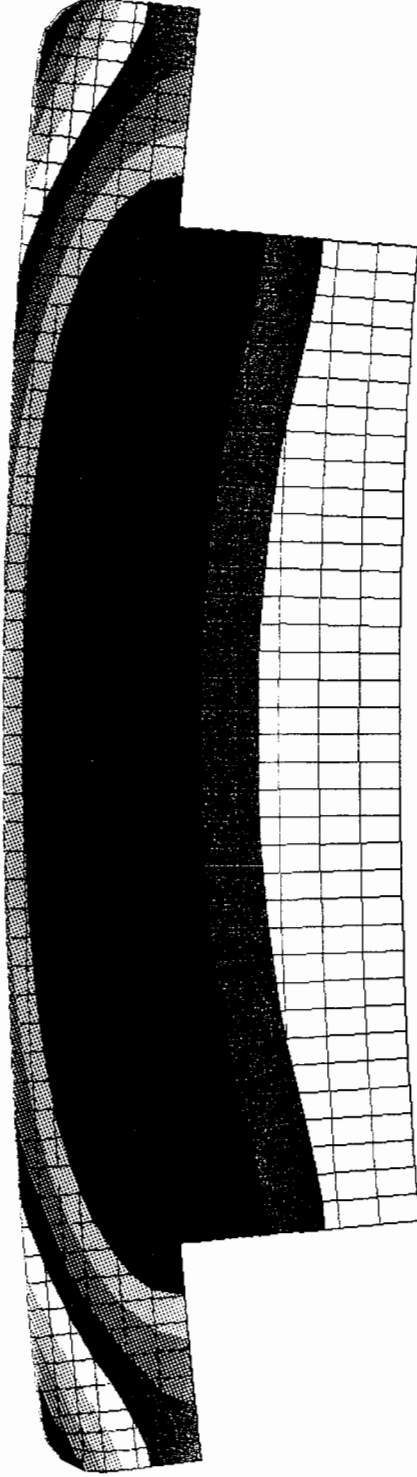
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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**2

1208

1174

1140

1107

1073

1039

1005

972

938

904

870

837

803

769

736

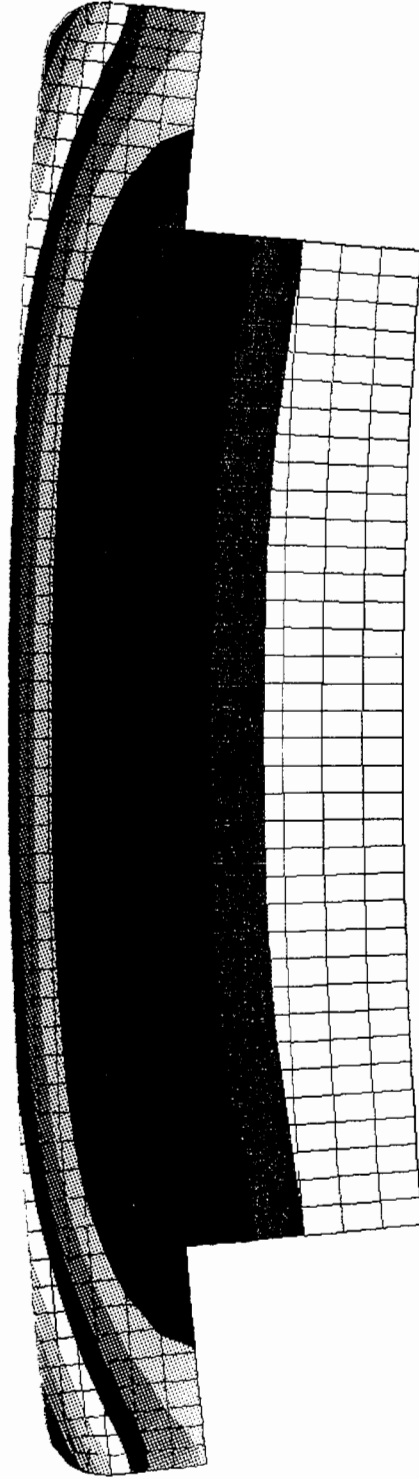
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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C
Qin = 320 Watt /cm**2

Unit = deg C

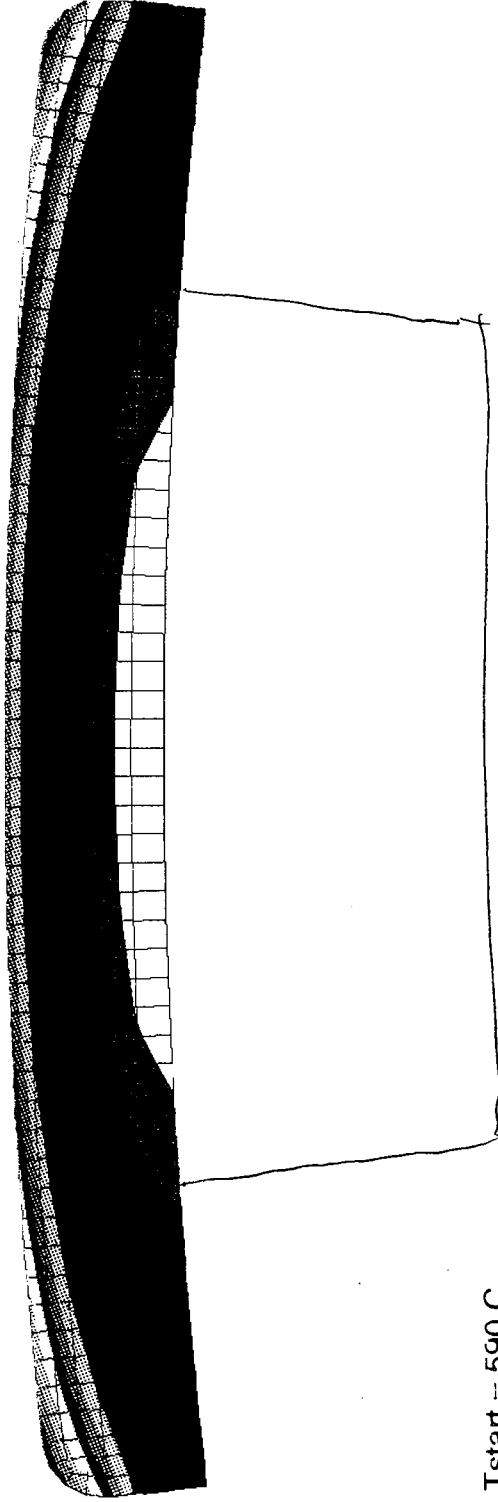
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955
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907
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859
835
811
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690
666
642
618

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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**2

Handwritten: 1.135

Unit = deg C

1864.

1849.

1834.

1819.

1805.

1790.

1775.

1761.

1746.

1731.

1716.

1702.

1687.

1672.

1658.

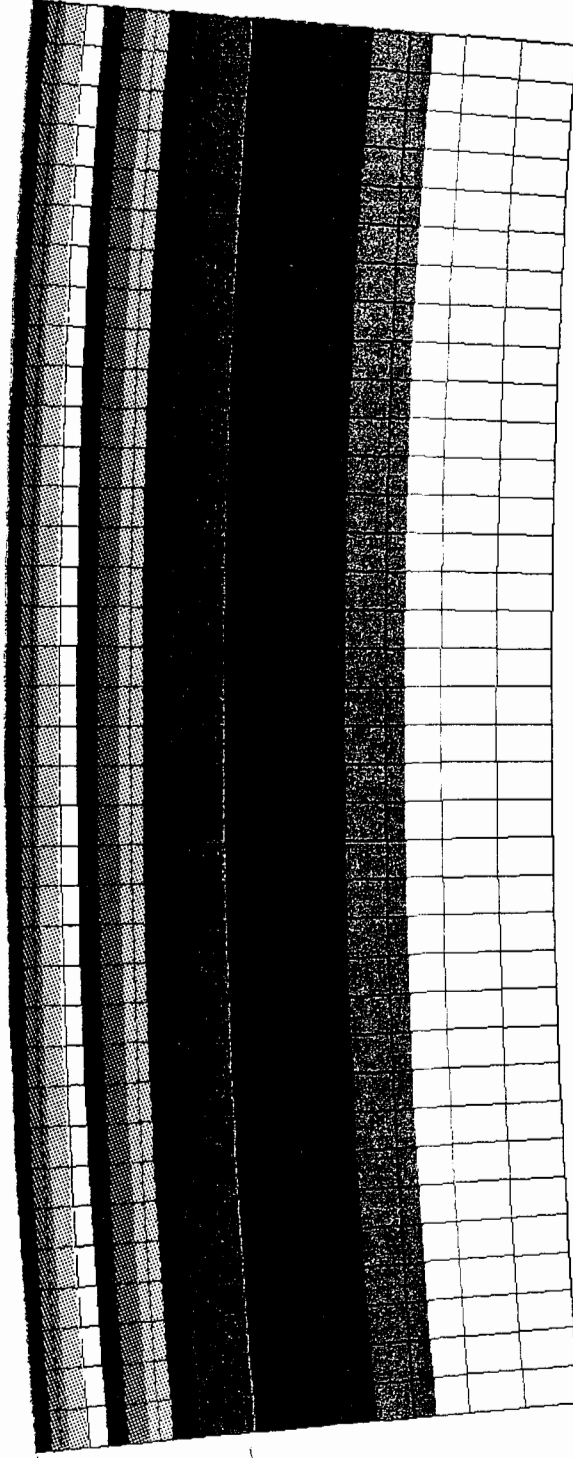
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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**2

$\epsilon = 0.8$

1314

1291

1267

1244

1221

1197

1174

1151

1127

1104

1080

1057

1034

1010

987

Unit = deg C

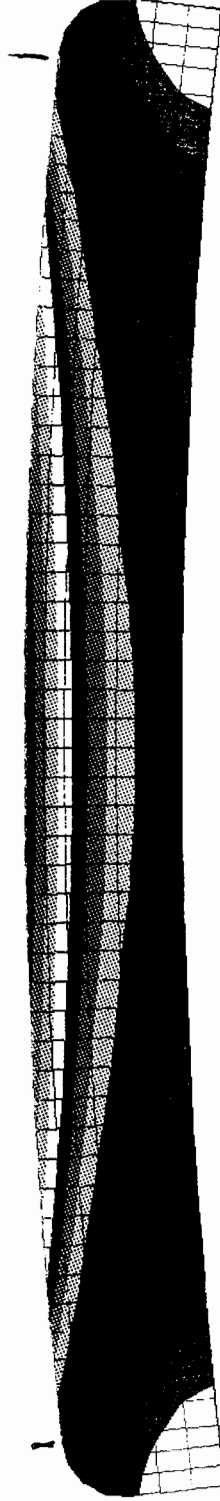
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NSTX Graphite Bumper Tile

Thermal Analysis



Tstart = 590 C

Qin = 320 Watt /cm**2

1764.

1745.

1727.

1708.

1690.

1671.

1653.

1634.

1616.

1597.

1579.

1560.

1542.

1523.

1505.

Unit = deg C

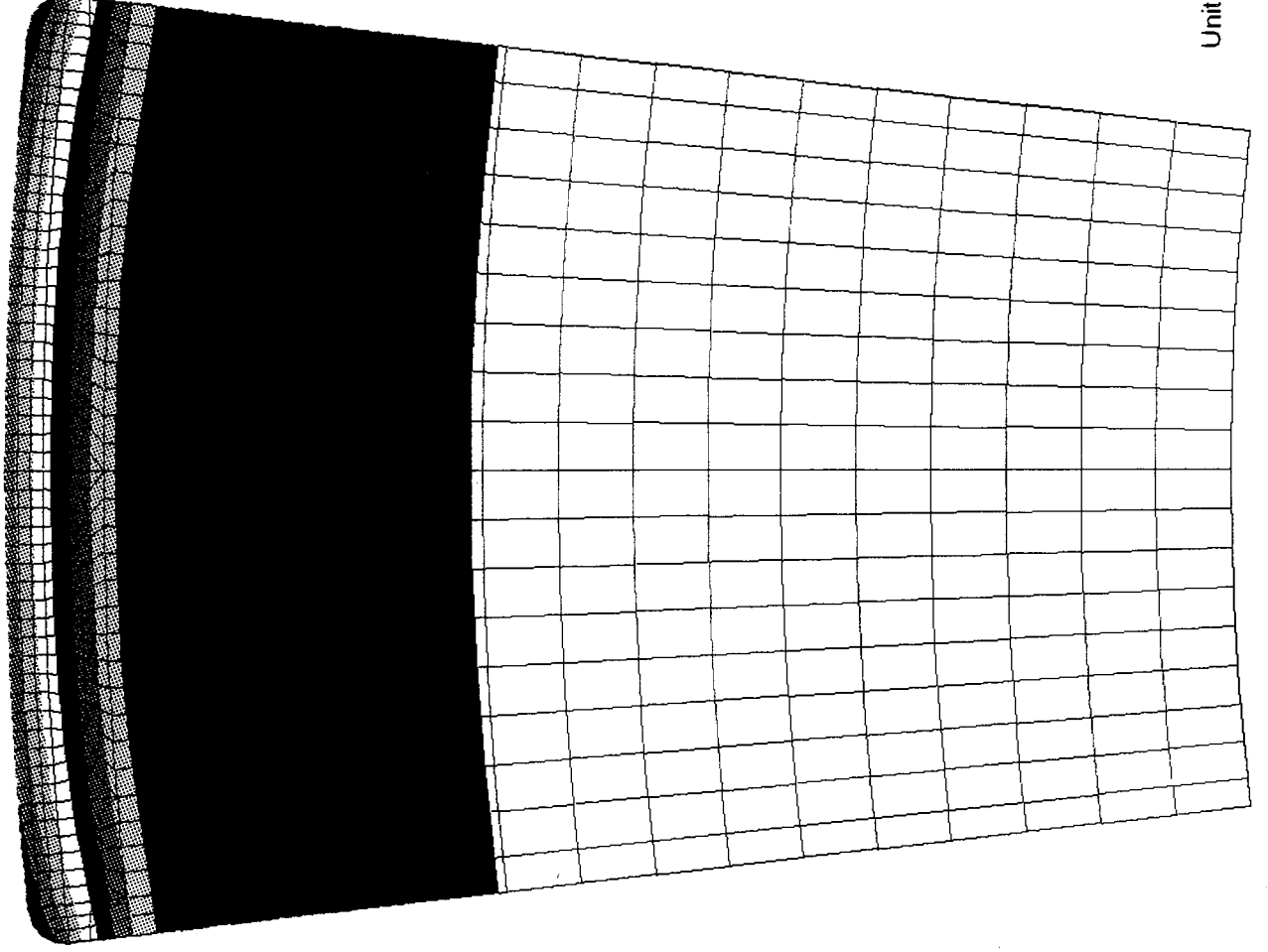
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NSTX Graphite Tile

Thermal Analysis
2 Inch Thick Tile



$Q_{in} = 320 \text{ Watt /cm}^{**2}$

Unit = deg C

1383

1335

1288

1241

1194

1147

1100

1052

1005

958

911

864

816

769

722

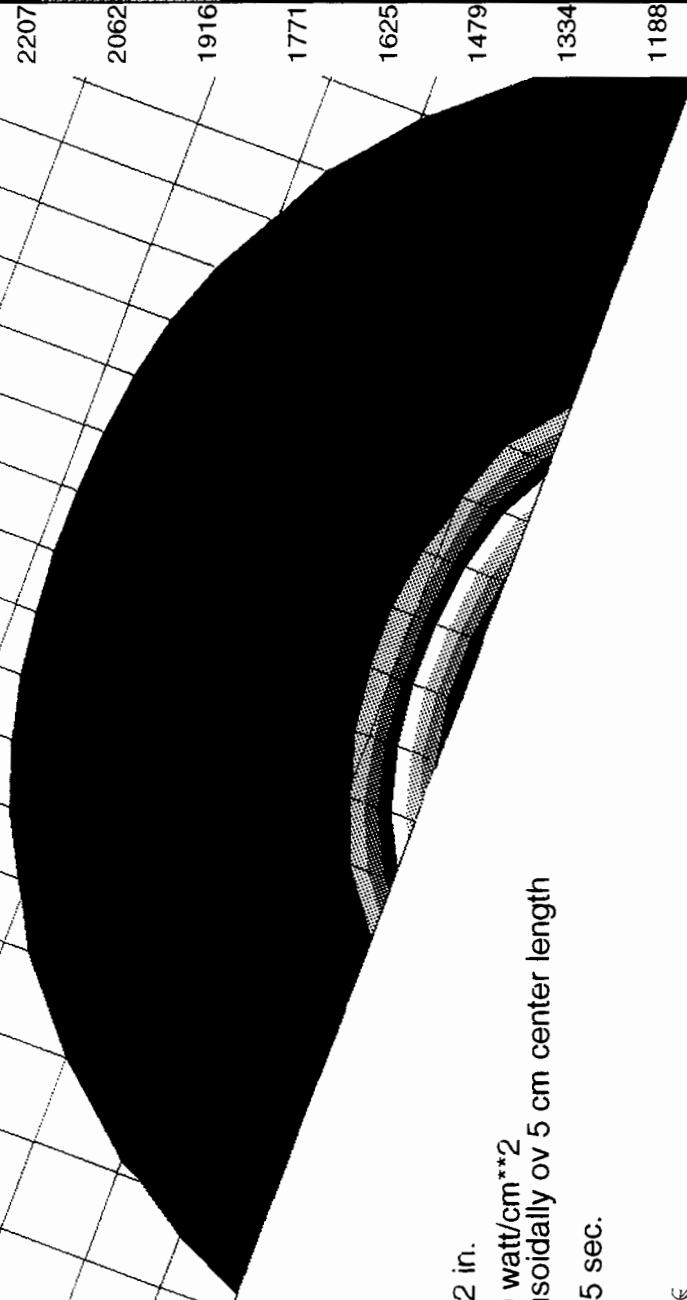
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NSTX Graphite Diverter Tile

Preliminary Thermal Analysis



Tile Thickness = 2 in.

Peak flux = 1700 watt/cm**2

applied sinusoidally ov 5 cm center length

Pulse duration = 5 sec.

Tstart = 540C

TAFTEC 4TH PULSE

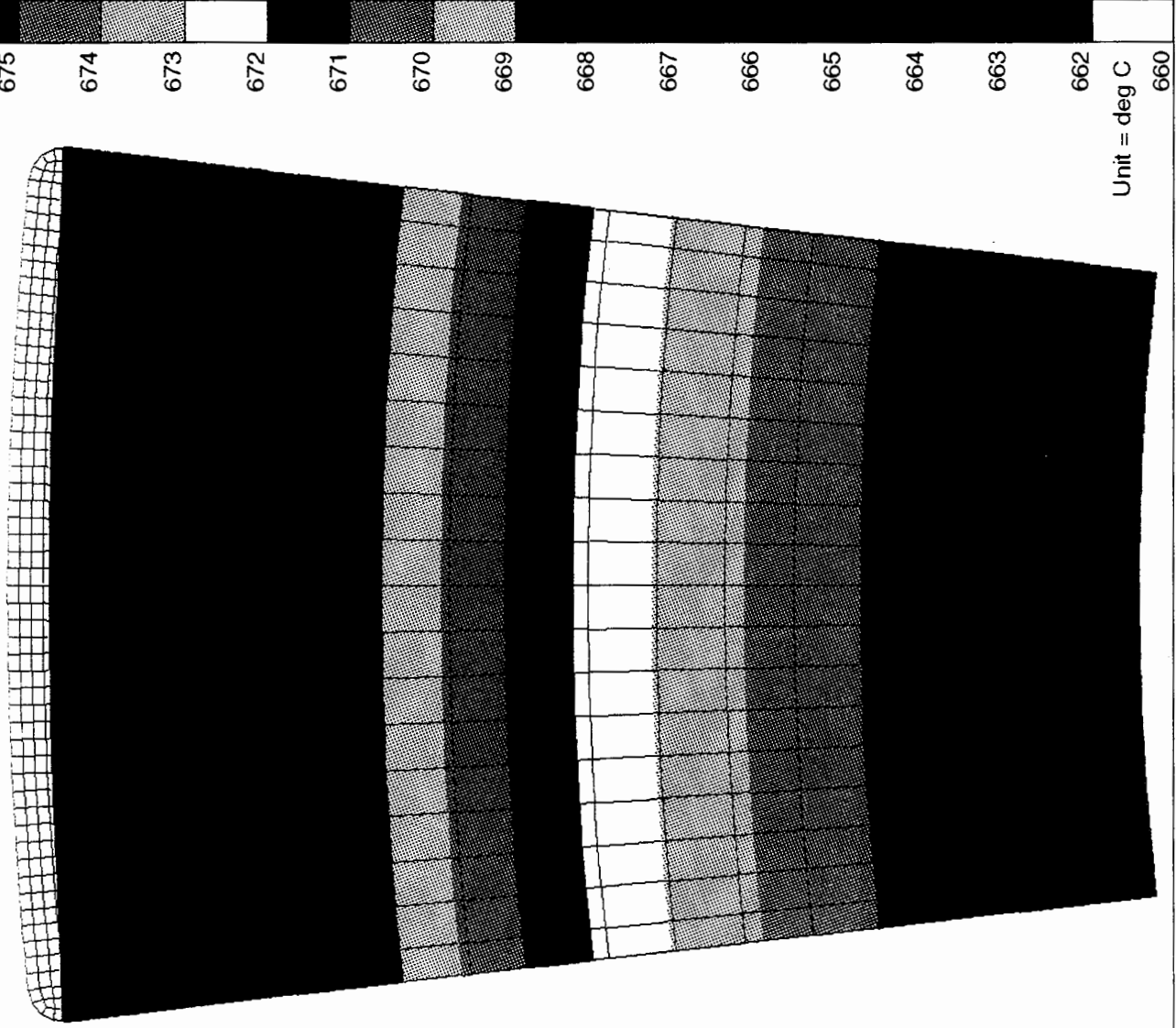
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2353
2207
2062
1916
1771
1625
1479
1334
1188
1043
897
752

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FRINGE: TIME: 8.9999000000D+02 SECONDS, nr60.nrf01: Temperature -PATRAN 2.5

NSTX Graphite Tile

Thermal Analysis
2 Inch Thick Tile



$Q_{in} = 320 \text{ Watt /cm}^2$

Unit = deg C

660

676

675

674

673

672

671

670

669

668

667

666

665

664

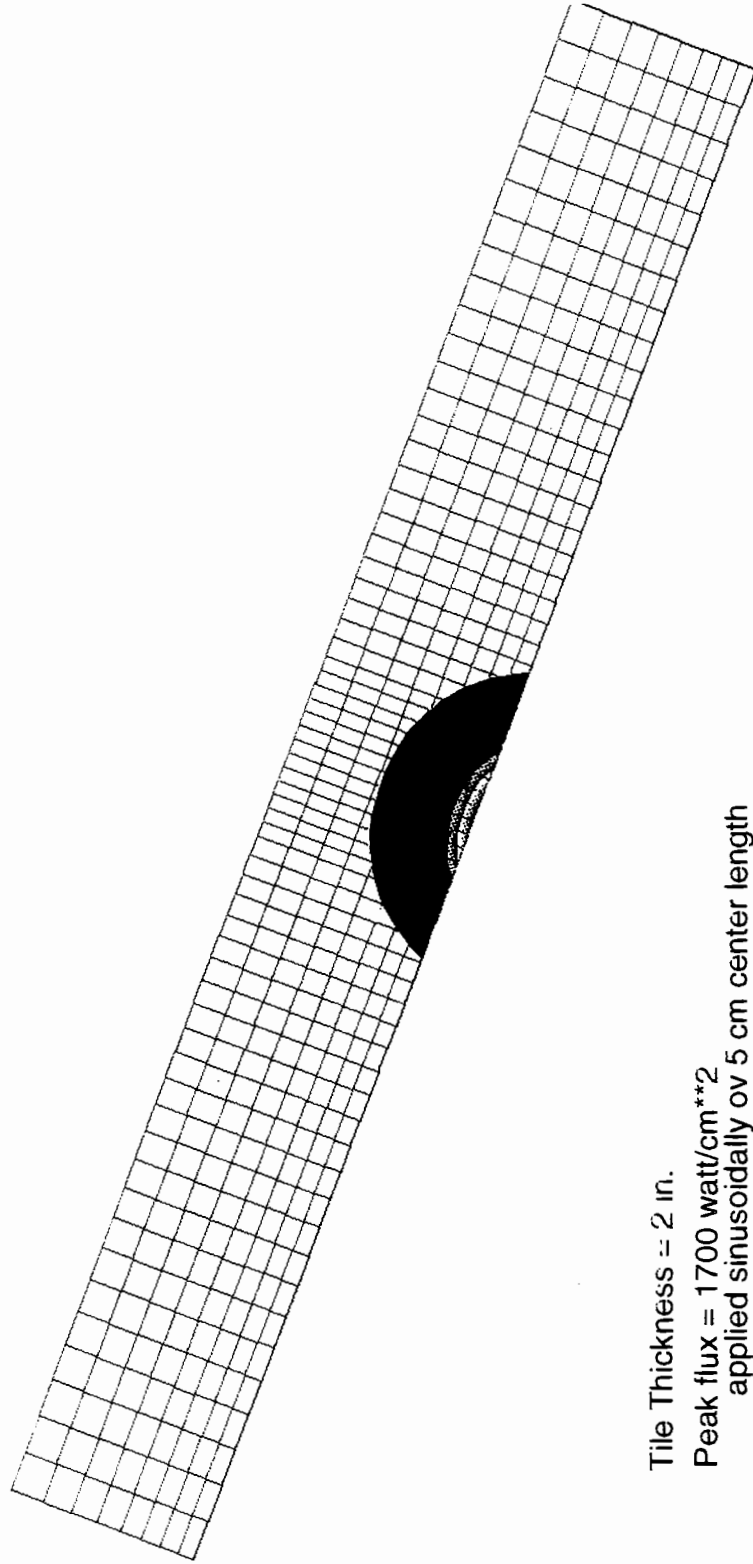
663

662

MSC/PATRAN Version 6.0 14-Apr-97 16:19:30
FRINGE: TIME: 9.050000000D+02 SECONDS, nr66.nrf01: Temperature -PATRAN 2.5

NSTX Graphite Diverter Tile

Preliminary Thermal Analysis



Tile Thickness = 2 in.

Peak flux = 1700 watt/cm**2
applied sinusoidally ov 5 cm center length

Pulse duration = 5 sec.

Tstart = 540C

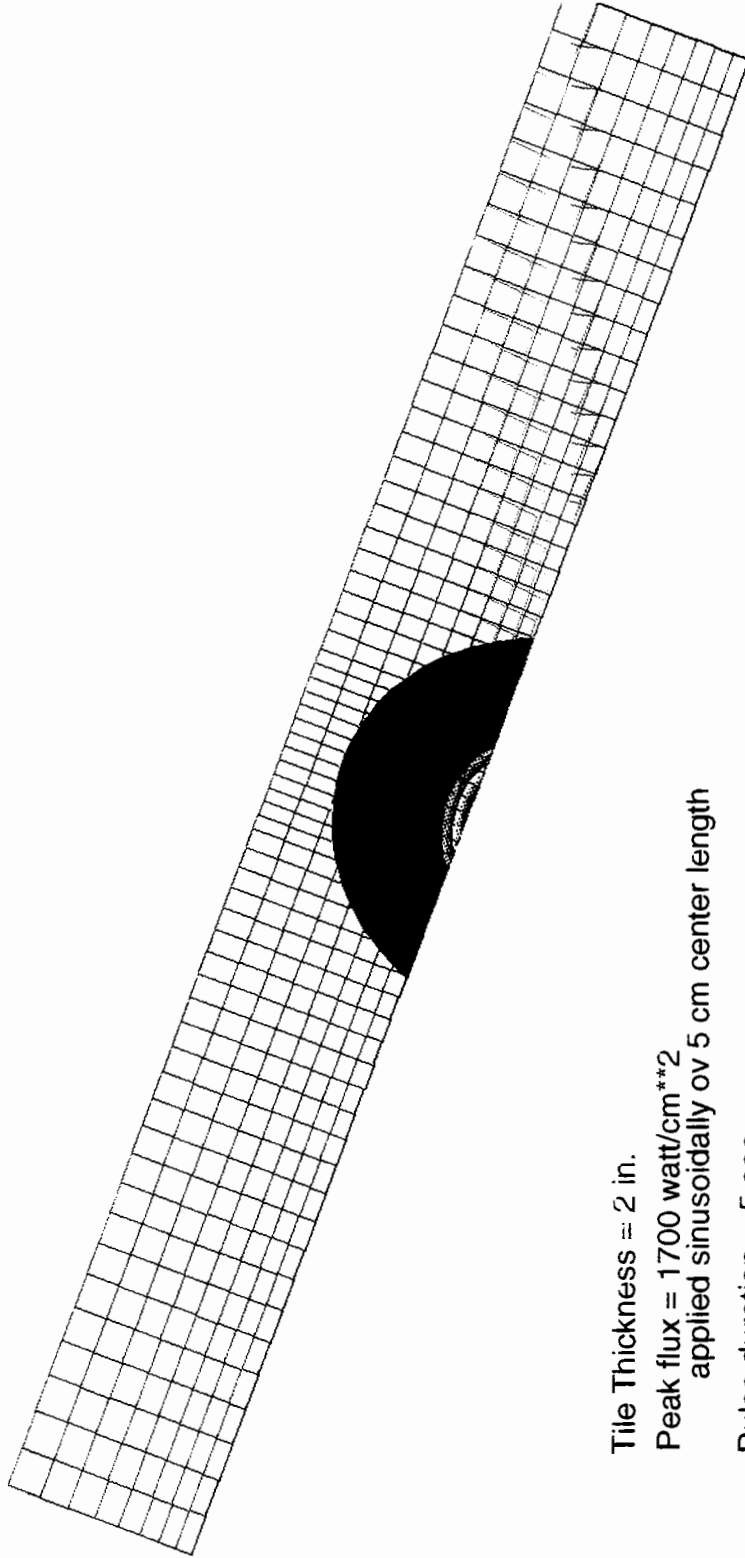
2935
2790
2644
2499
2353
2207
2062
1916
1771
1625
1479
1334
1188
1043
897
752

MSC/PATRAN Version 6.0 14-Apr-97 15:37:39

FRINGE: TIME: 5.000000000D+00 SECONDS, nr5.nrf01: Temperature -PATRAN 2.5

NSTX Graphite Diverter Tile

Preliminary Thermal Analysis



Tile Thickness = 2 in.

Peak flux = 1700 watt/cm**2
applied sinusoidally ov 5 cm center length

Pulse duration = 5 sec.

Tstart = ~~5400~~

0

Conversion = 2 X POCO

2487.

2341.

2194.

2048.

1902.

1755.

1609.

1463.

1316.

1170.

1023.

877.1

730.7

584.3

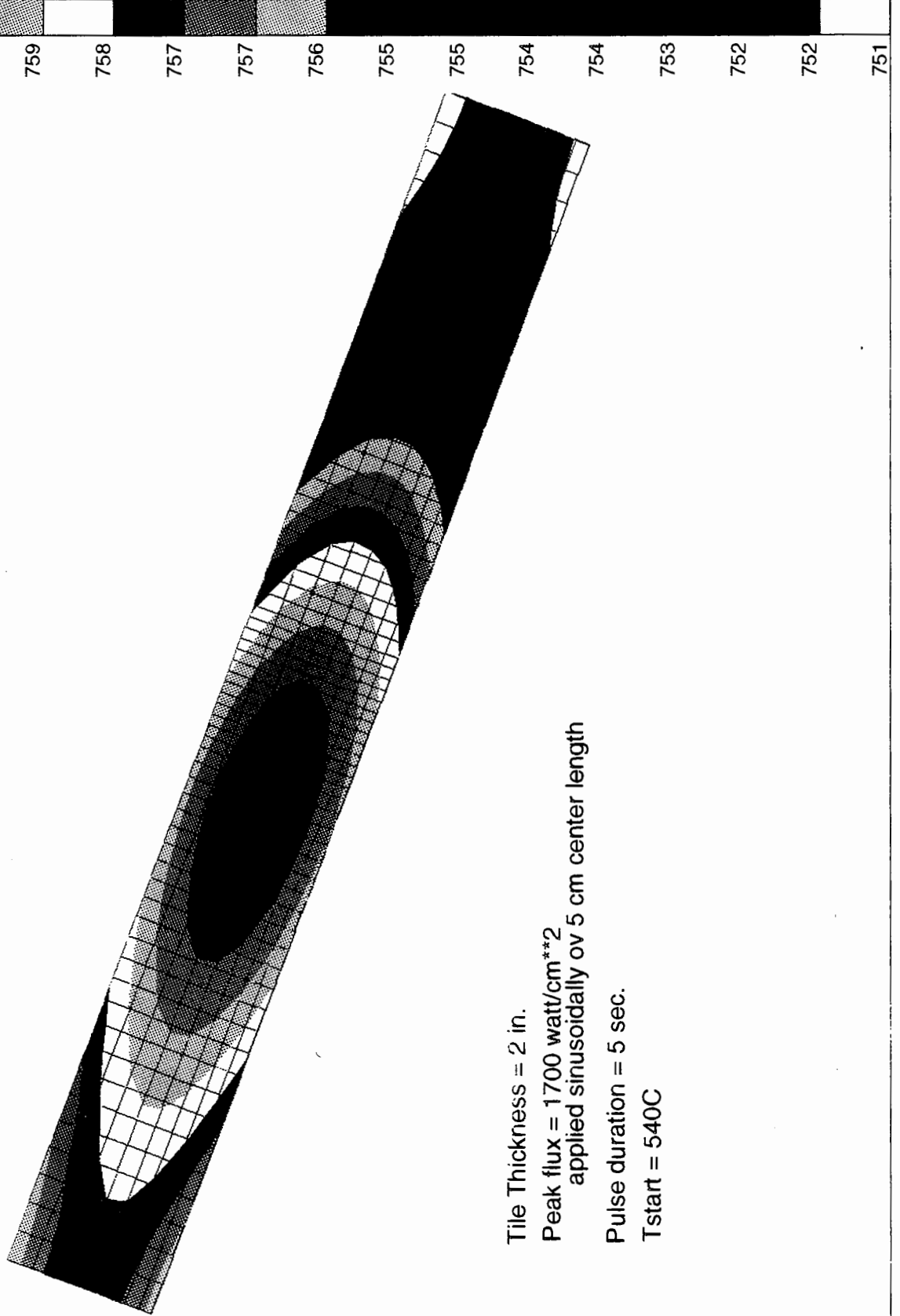
437.9

291.6

MSC/PATRAN Version 6.0 14-Apr-97 16:20:14
FRINGE: TIME: 8.9999000000D+02 SECONDS, nr60.nrf01: Temperature -PATRAN 2.5

NSTX Graphite Diverter Tile

Preliminary Thermal Analysis



Tile Thickness = 2 in.

Peak flux = 1700 watt/cm^2
applied sinusoidally ov 5 cm center length

Pulse duration = 5 sec.

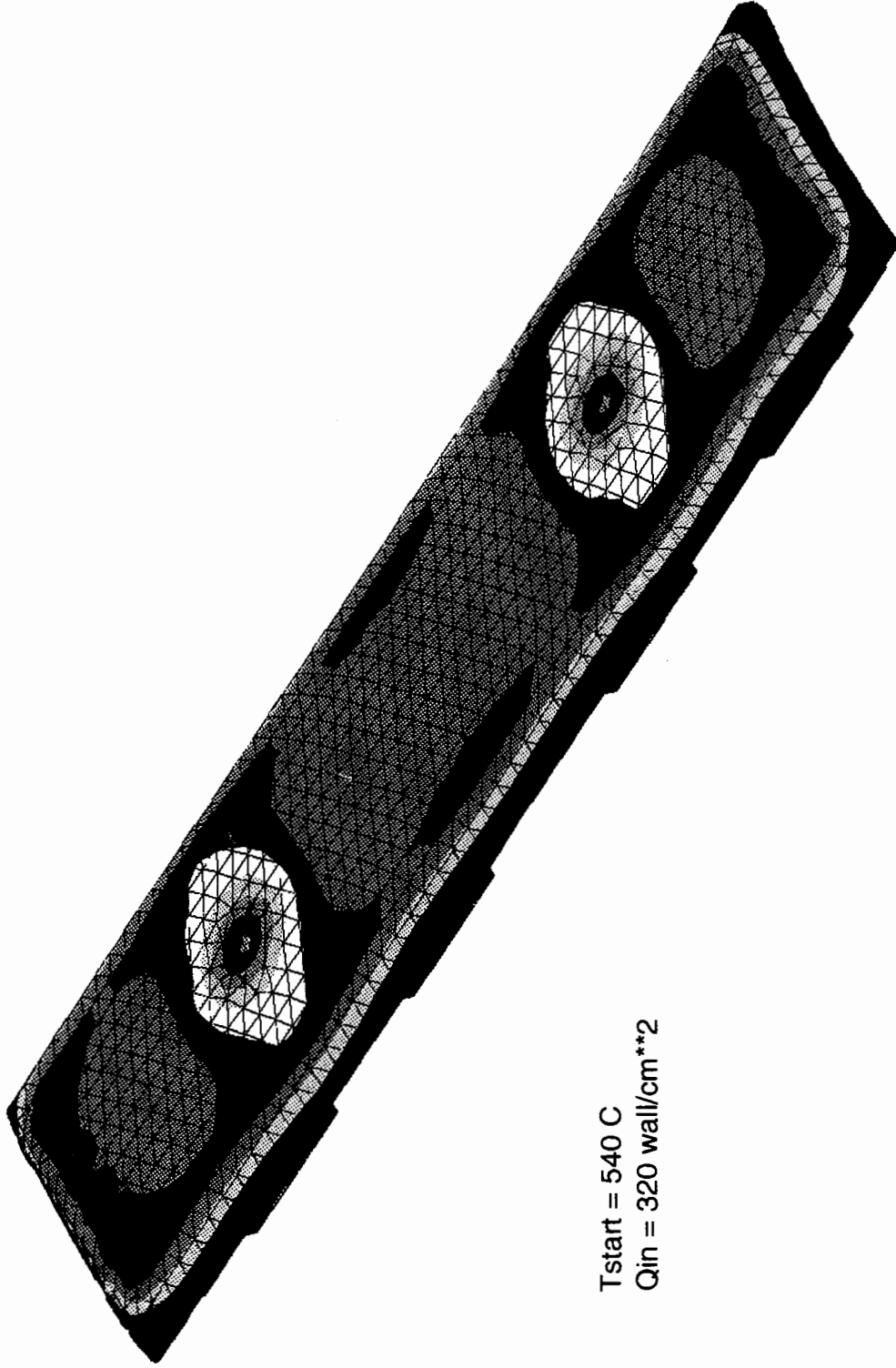
Tstart = 540C

MSC/PATRAN Version 6.0 25-Jul-97 14:44:09

FRINGE: 320_watts/cm**2, nr5.nrf01: Temperature -PATRAN 2.5

NSTX Center Stack Graphite Tile

Thermal Analysis



Tstart = 540 C
Qin = 320 wall/cm**2

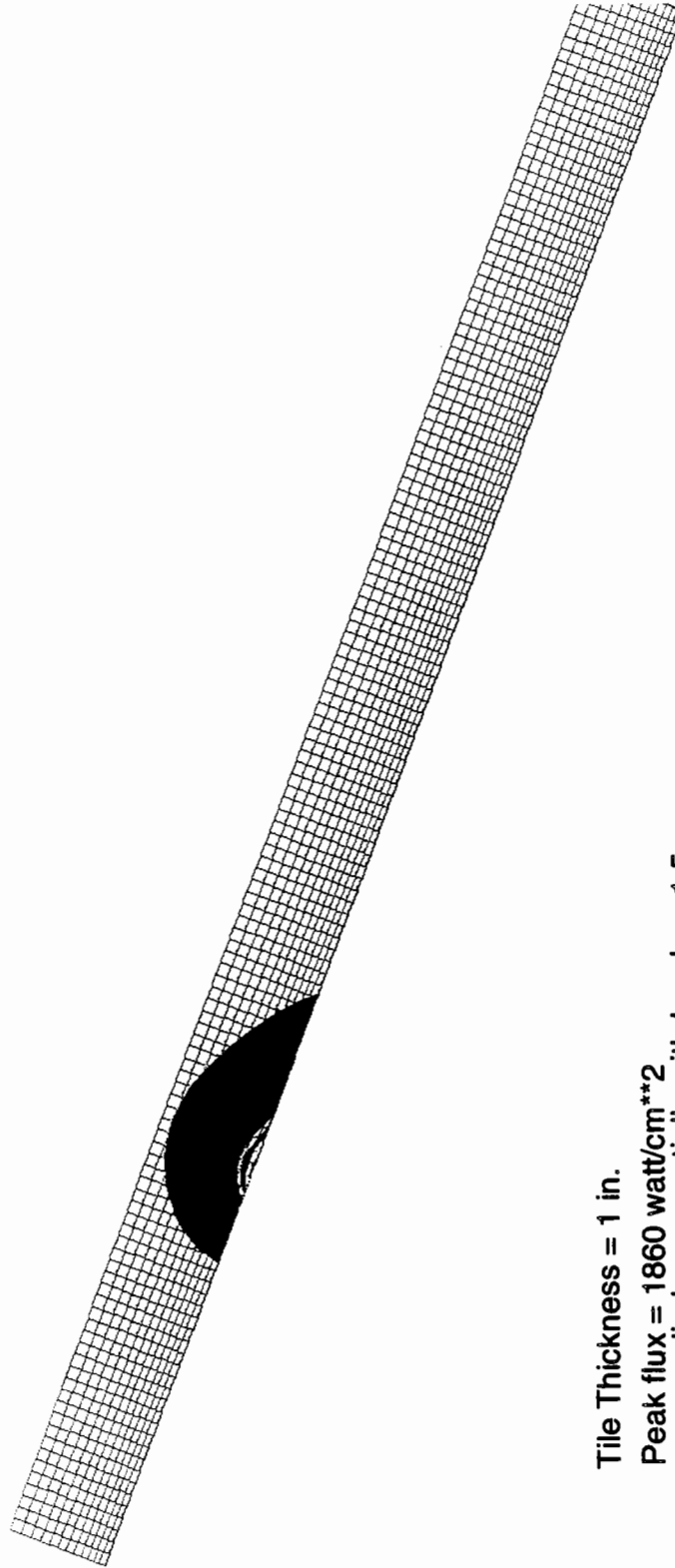
Temp Unit = deg C

MSC/PATRAN Version 7.0 04-Dec-97 12:16:27

FRINGE: Tmax, ATJ, 5sec, lamda = 1.5, ATJ_5sec_lamda_1.5nrf: Temperature -PATRAN 2.5

NSTX Graphite Divertor Tile

Preliminary Thermal Analysis



Tile Thickness = 1 in.

Peak flux = 1860 watt/cm**2
applied exponentially with lamda = 1.5 cm

Pulse duration = 5 sec.

Tstart = 150C

2602

2439

2275

2112

1948

1785

1621

1458

1294

1131

967

804

640

476

313

Unit = deg C

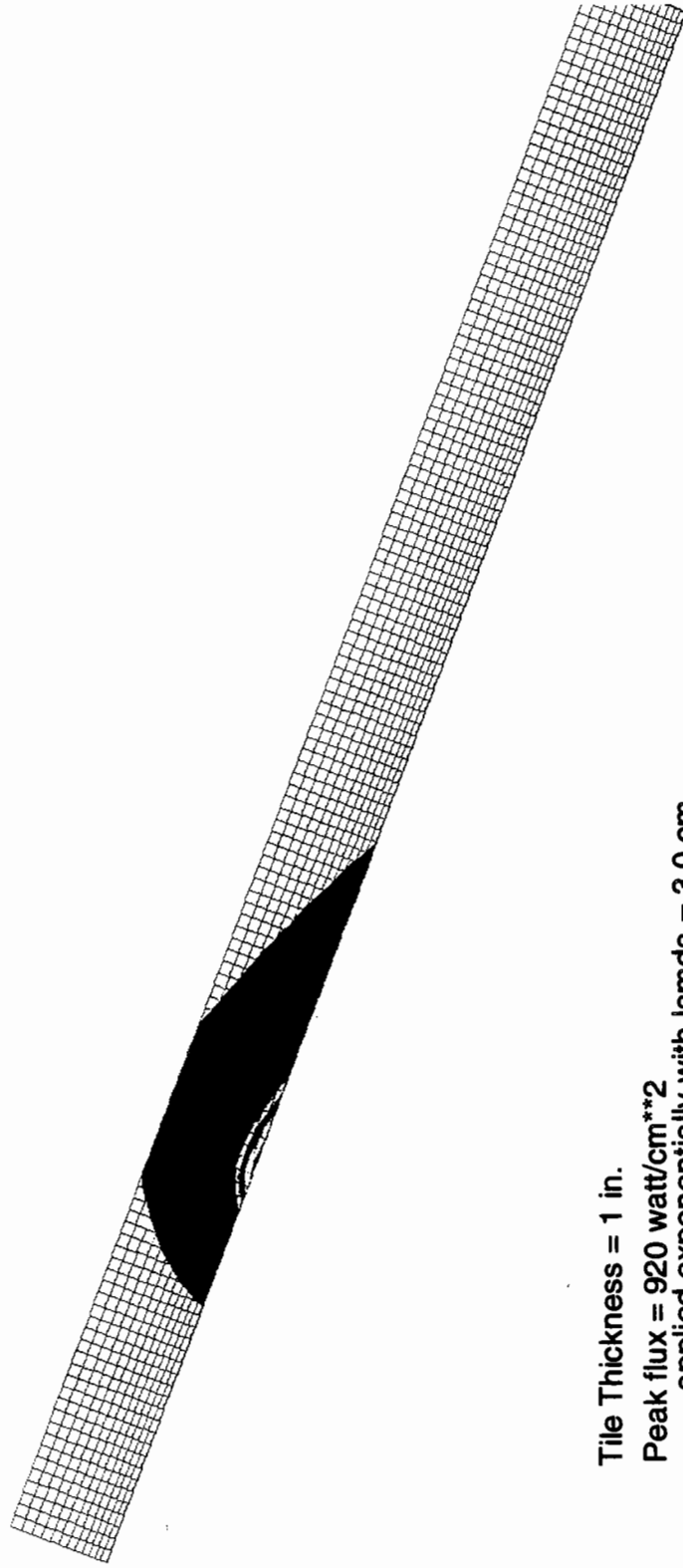
149

MSC/PATRAN Version 7.0 04-Dec-97 12:10:05

FRINGE: Tmax, ATJ, 5sec, lamda = 3.0, ATJ_5sec_lamda_3nrf: Temperature -PATRAN 2.5

NSTX Graphite Divertor Tile

Preliminary Thermal Analysis



Tile Thickness = 1 in.

Peak flux = 920 watt/cm**2
applied exponentially with lamda = 3.0 cm

Pulse duration = 5 sec.

Tstart = 150C

1532

1440

1348

1256

1164

1071

979

887

795

703

610

518

426

334

242

Unit = deg C

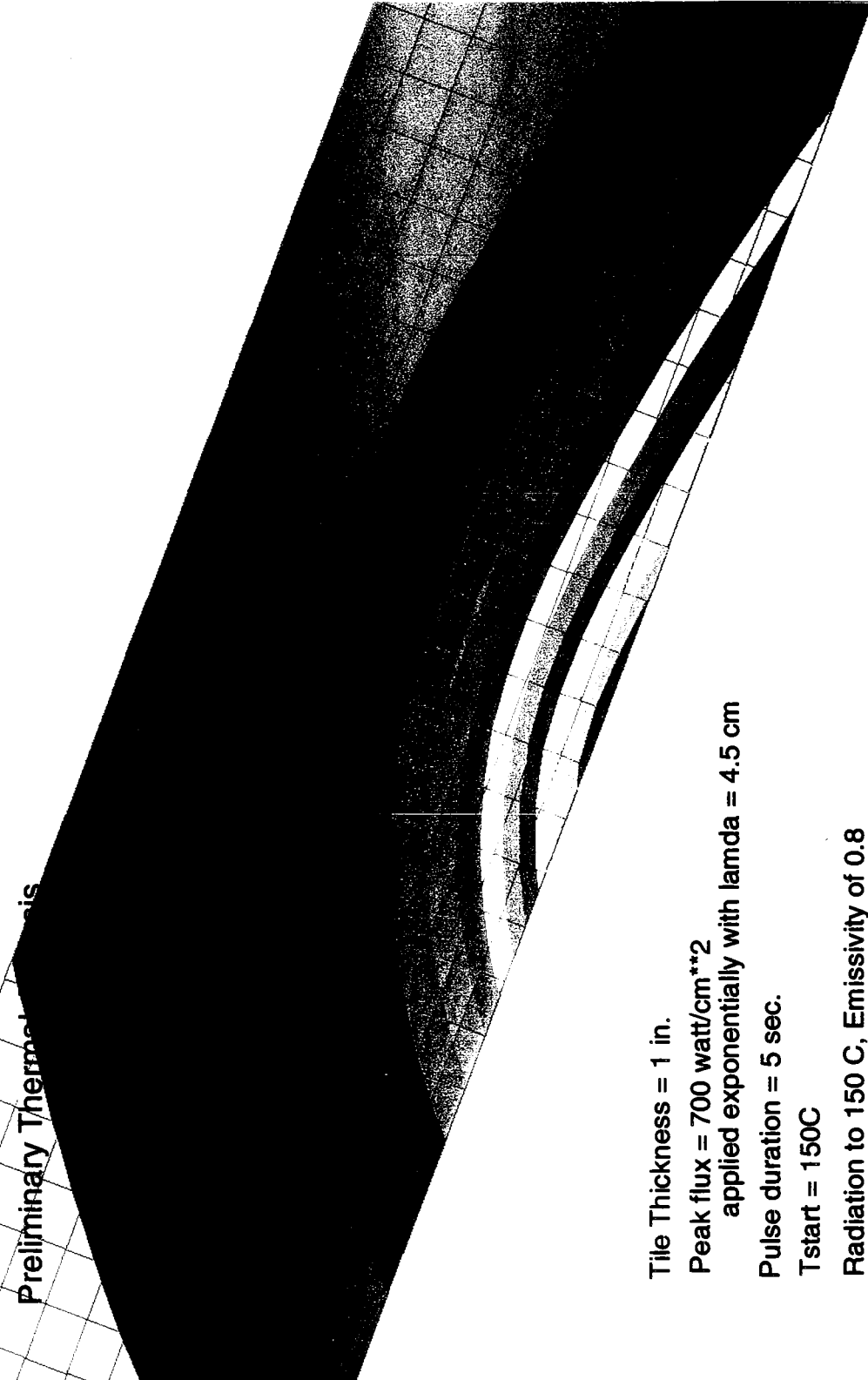
149

MSC/PATRAN Version 7.0 09-Dec-97 12:49:33

FRINGE: Tmax, ATJ, 5sec, lamda = 4.5, ATJ_5sec_lamda_045nrf: Temperature -PATRAN 2.5

NSTX Graphite Divertor Tile

Preliminary Thermal Analysis



Tile Thickness = 1 in.
Peak flux = 700 watt/cm**2
applied exponentially with lamda = 4.5 cm
Pulse duration = 5 sec.
Tstart = 150C
Radiation to 150 C, Emissivity of 0.8

1127
1061
996
931
866
801
736
671
606
541
475
410
345
280
215
150

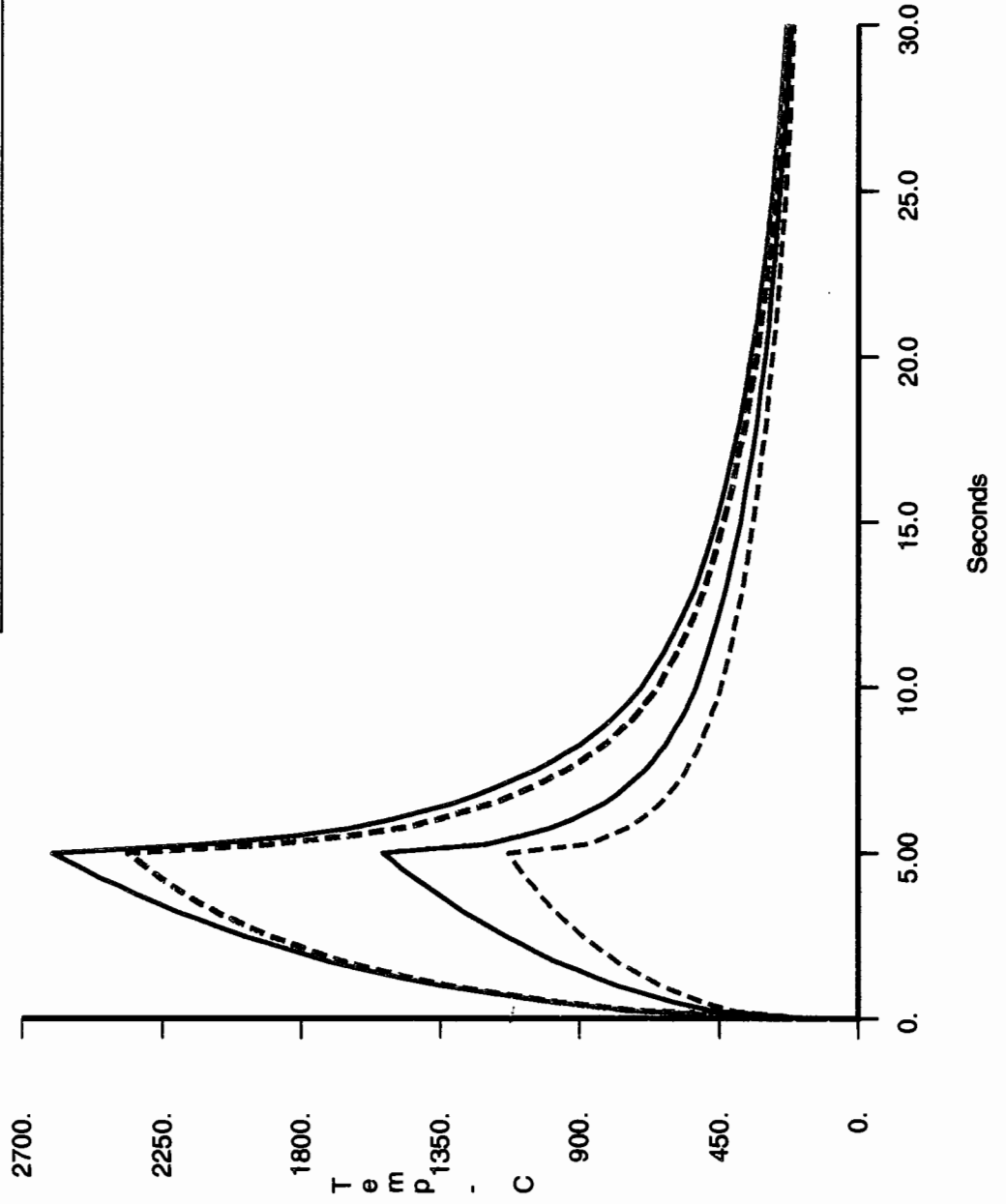
Unit = deg C

NSTX Divertor Tiles

ATJ Graphite
1 inch thickness

LEGEND

- Node 1263, Lamda = 1.5 cm
- - - Node 1263, Lamda = 1.5 cm, with Radiation
- Node 1263, Lamda = 3.0
- - - Node 1263, Lamda = 4.5

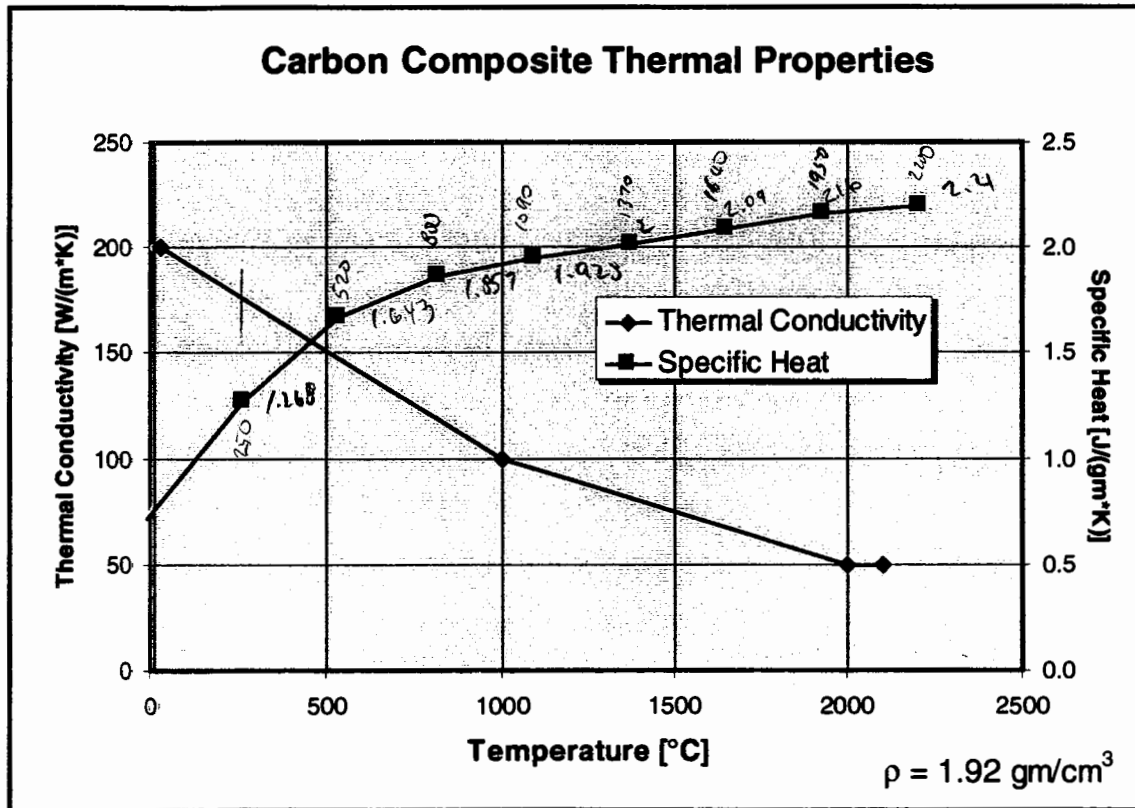


NSTX Inner Stack Carbon Tile Thermal Analysis

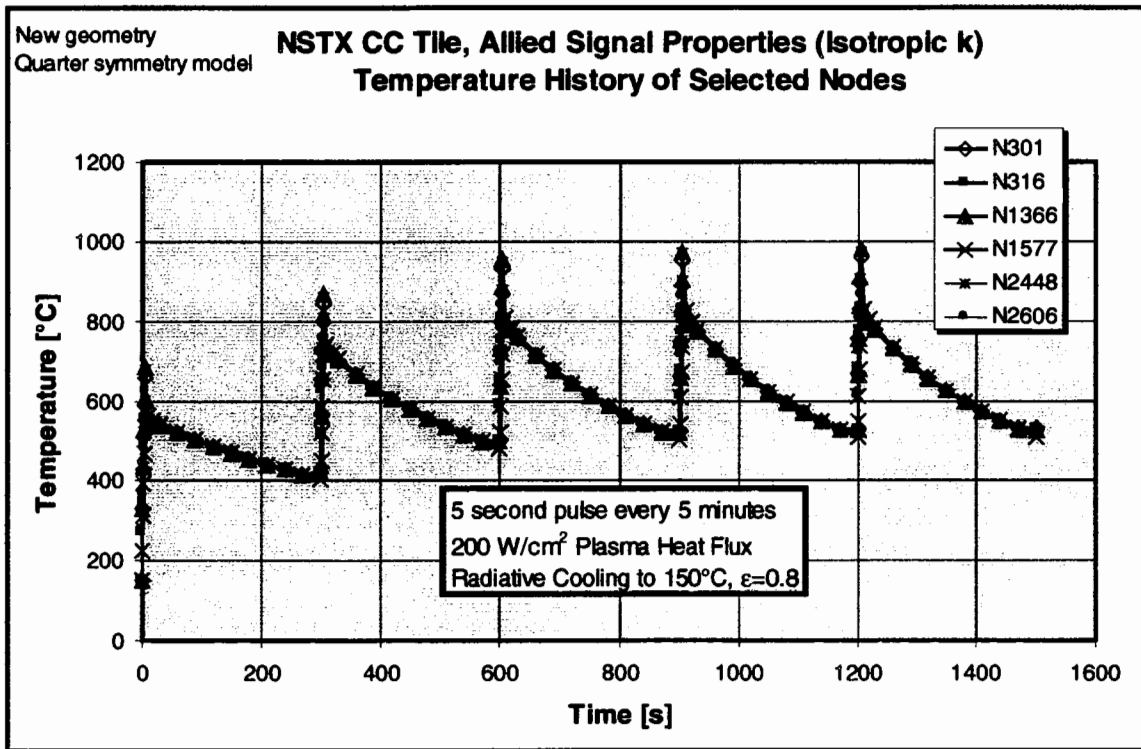
- Transient simulation: 5 second pulse every 5 minutes.
- Plasma heat flux: 200 W/cm²
- Radiative cooling to 150°C sink, $\epsilon = 0.8$

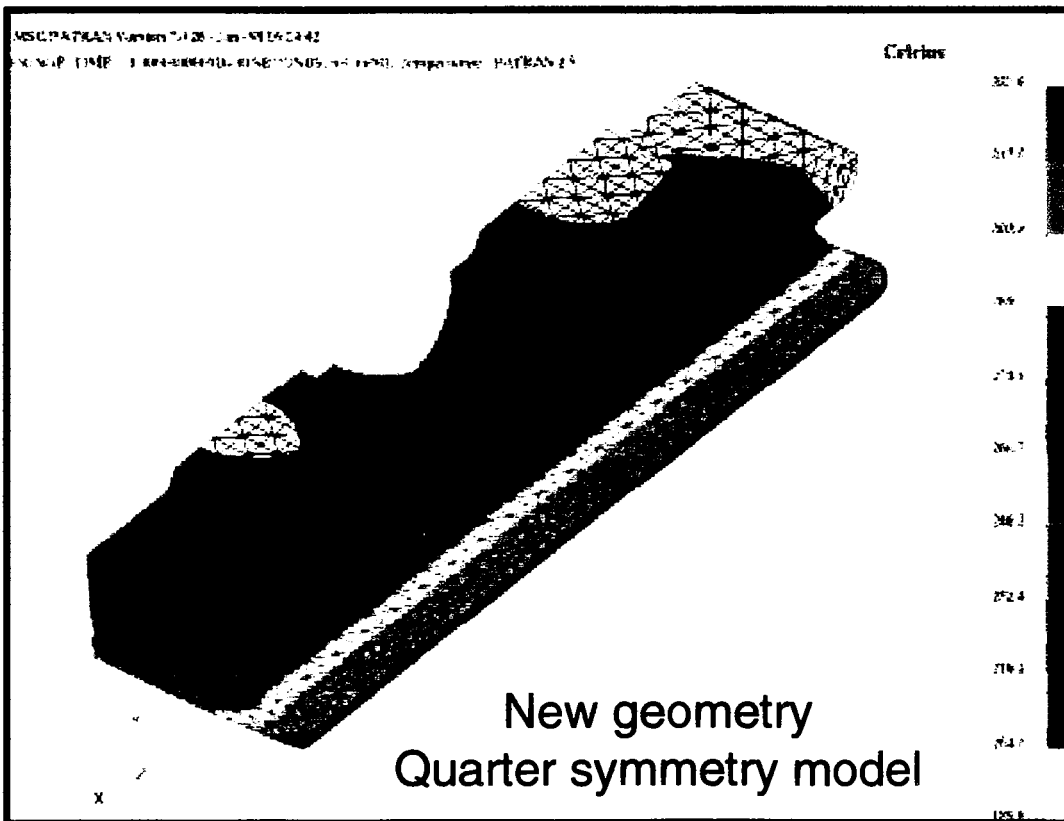
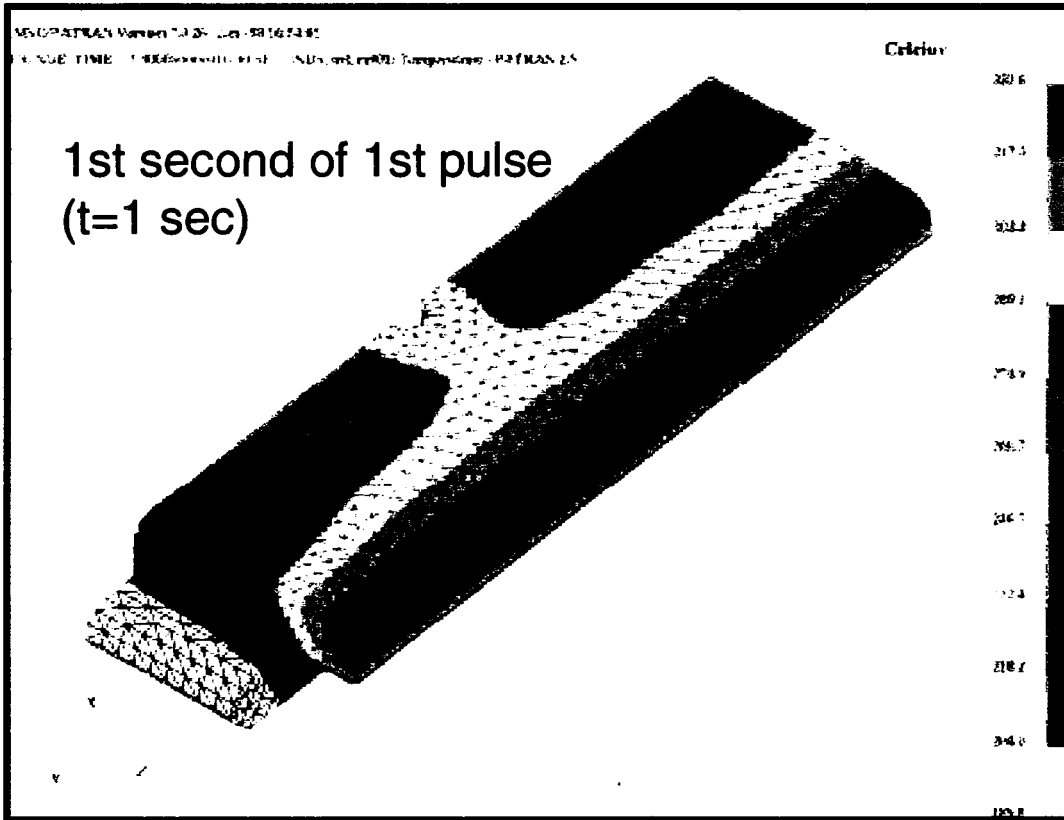
- Allied Signal carbon composite material.
- Insufficient data for orthotropic thermal conductivity; employed isotropic thermal properties.

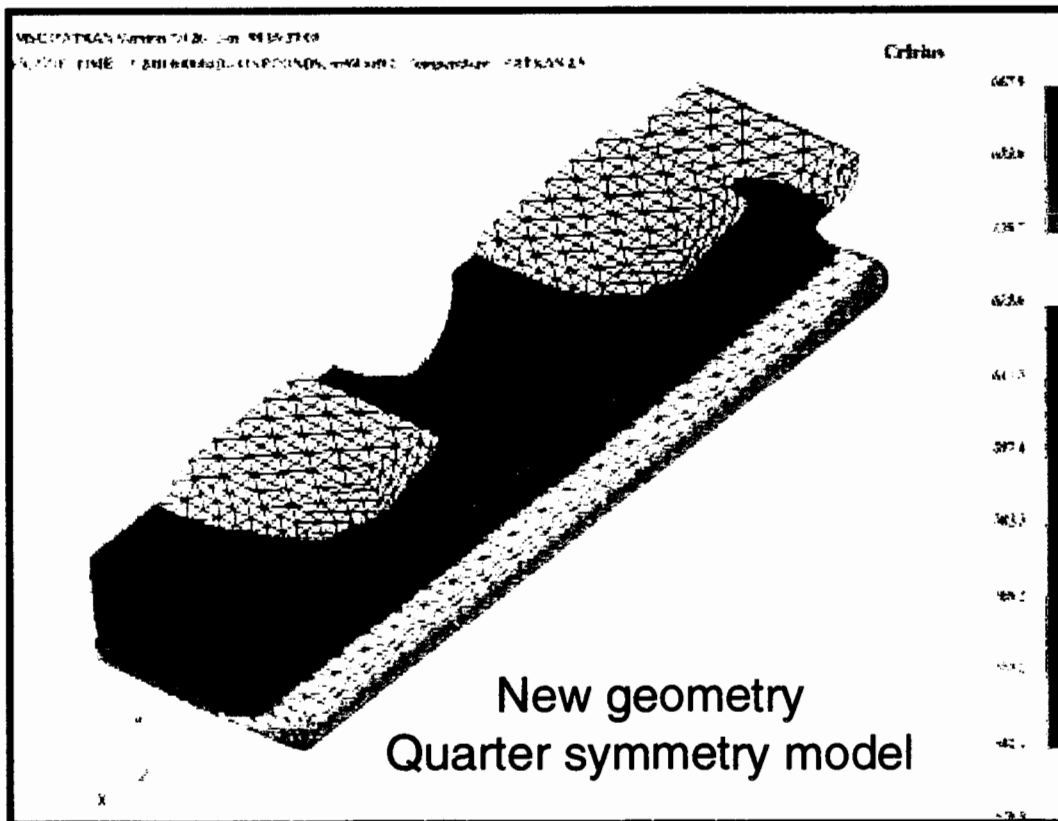
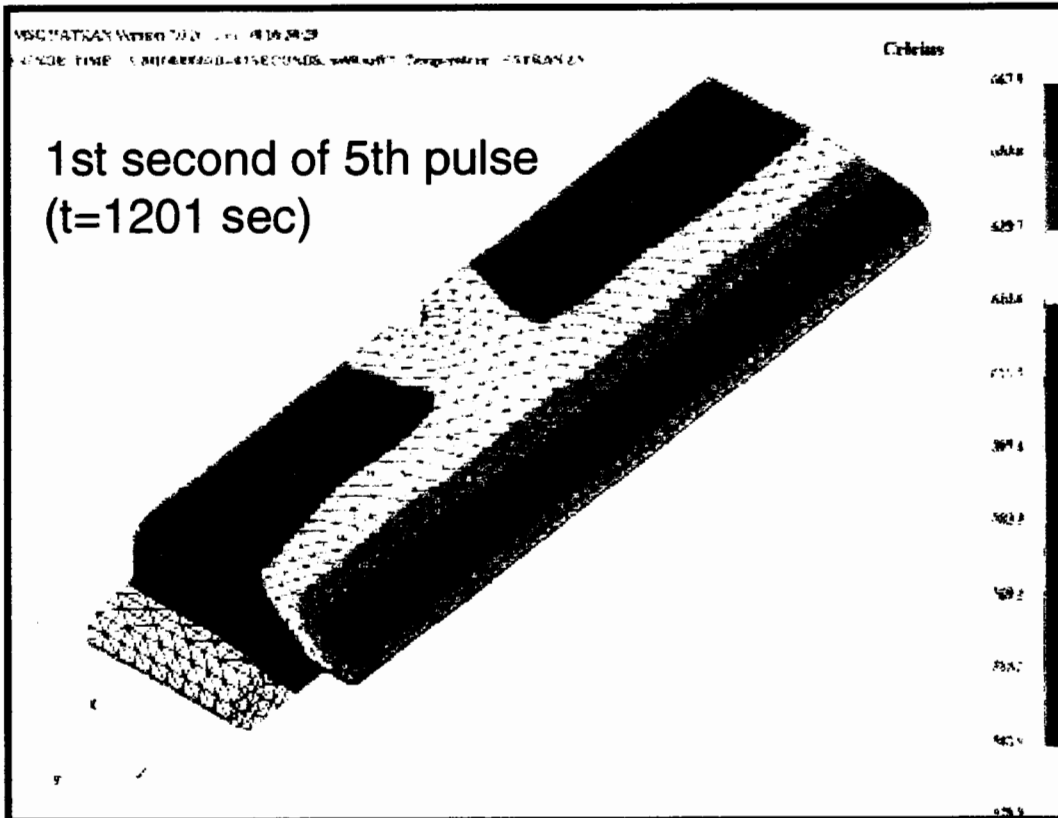
- Quarter symmetry model using recent ProE geometry (Jan. 25.)

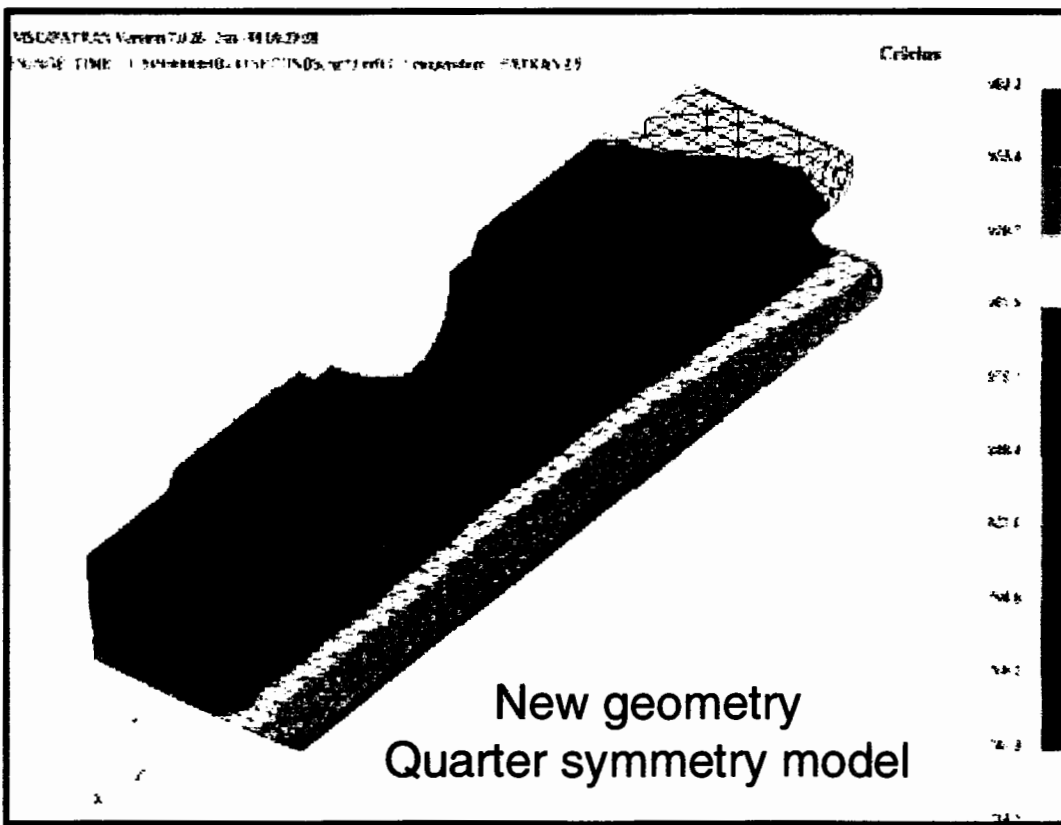
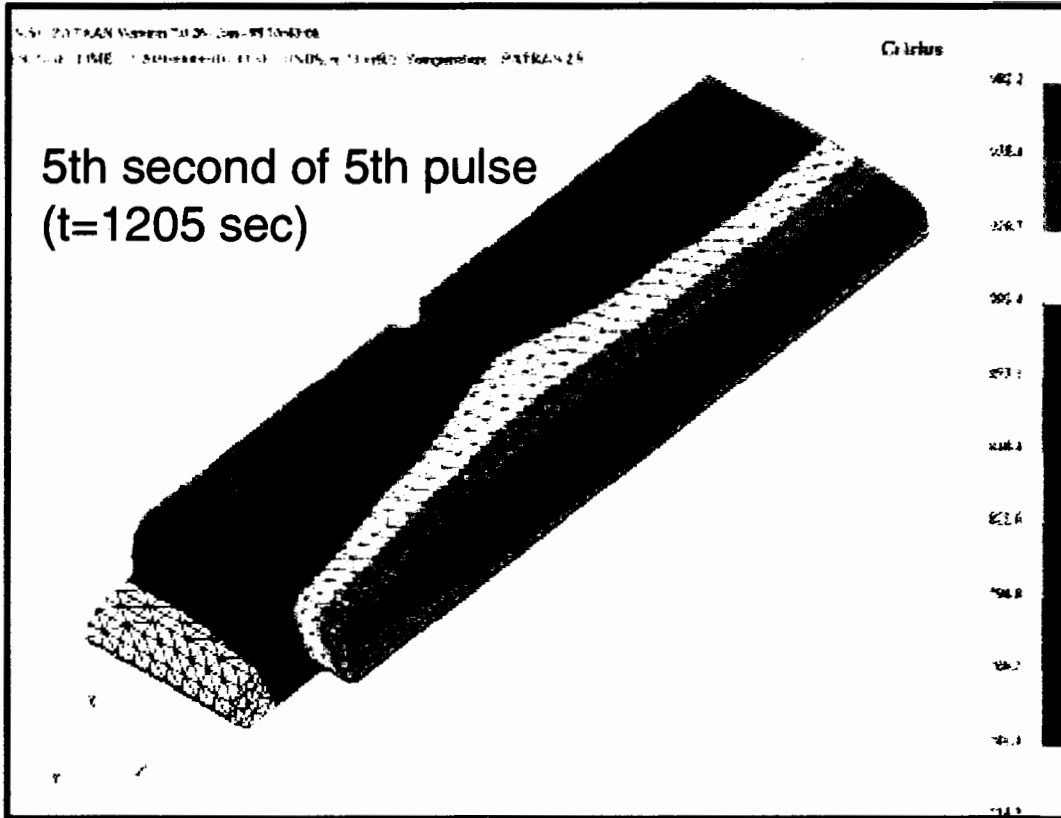


- Maximum temperature at end of 5th pulse is 982°C.
- Pre-pulse temperature after 5 cycles is < 530°C.
Thermal ratcheting nearly leveled off.



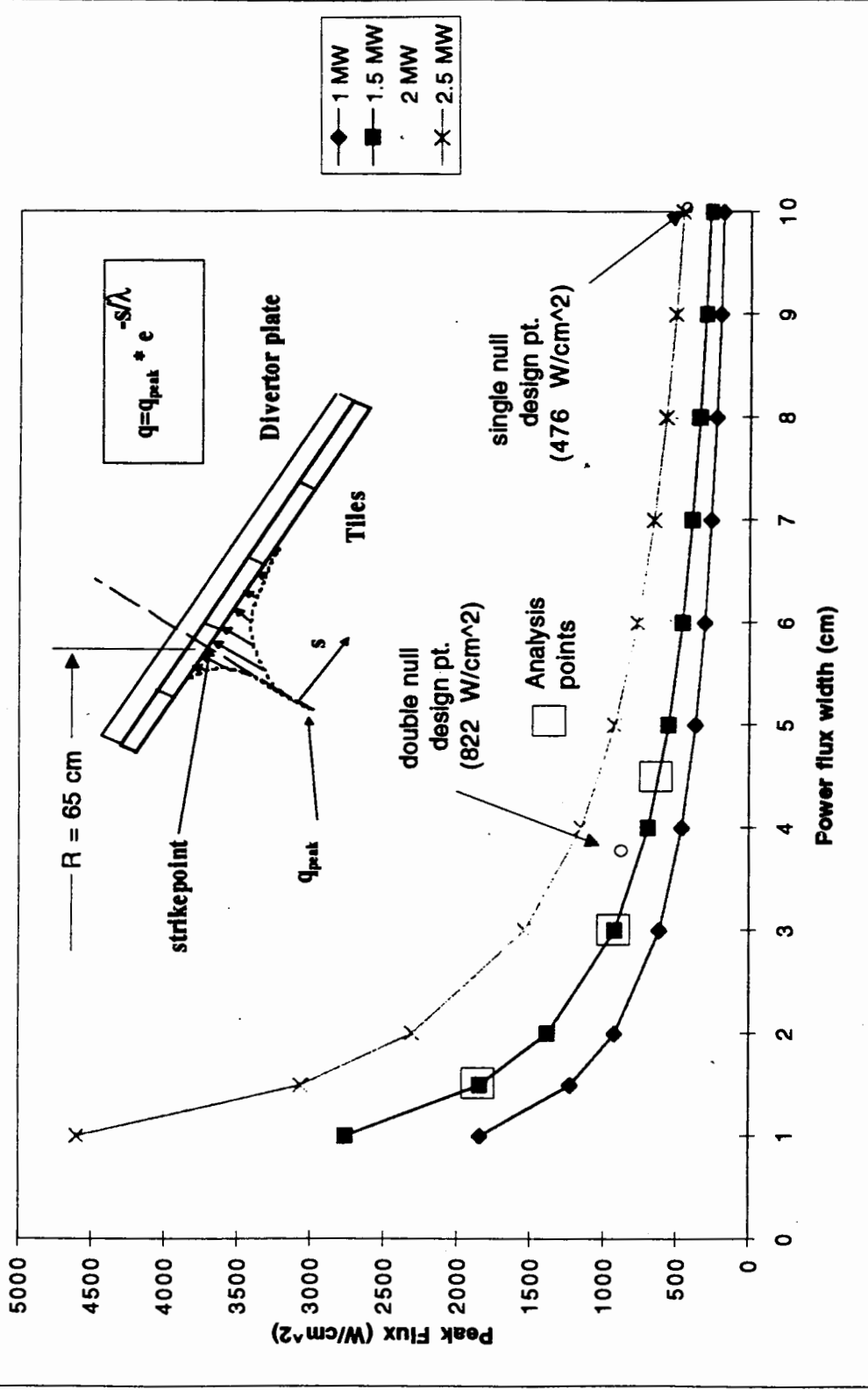






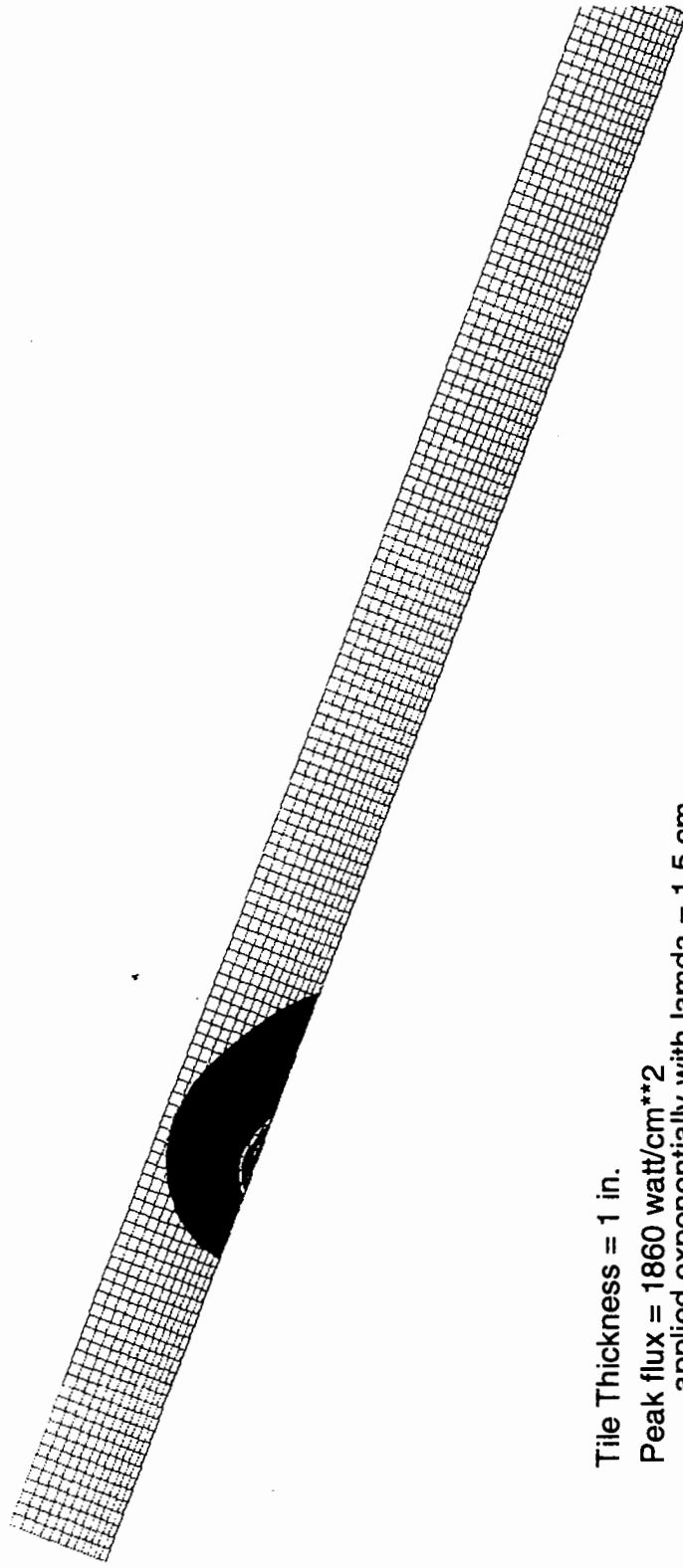
Peak Flux vs Power flux width, infinite distr.

(Radius to strike point = 65 cm)



NSTX Graphite Divertor Tile

Preliminary Thermal Analysis



Tile Thickness = 1 in.

Peak flux = 1860 watt/cm**2
applied exponentially with lamda = 1.5 cm

Pulse duration = 5 sec.

Tstart = 150C

2602

2439

2275

2112

1948

1785

1621

1458

1294

1131

967

804

640

476

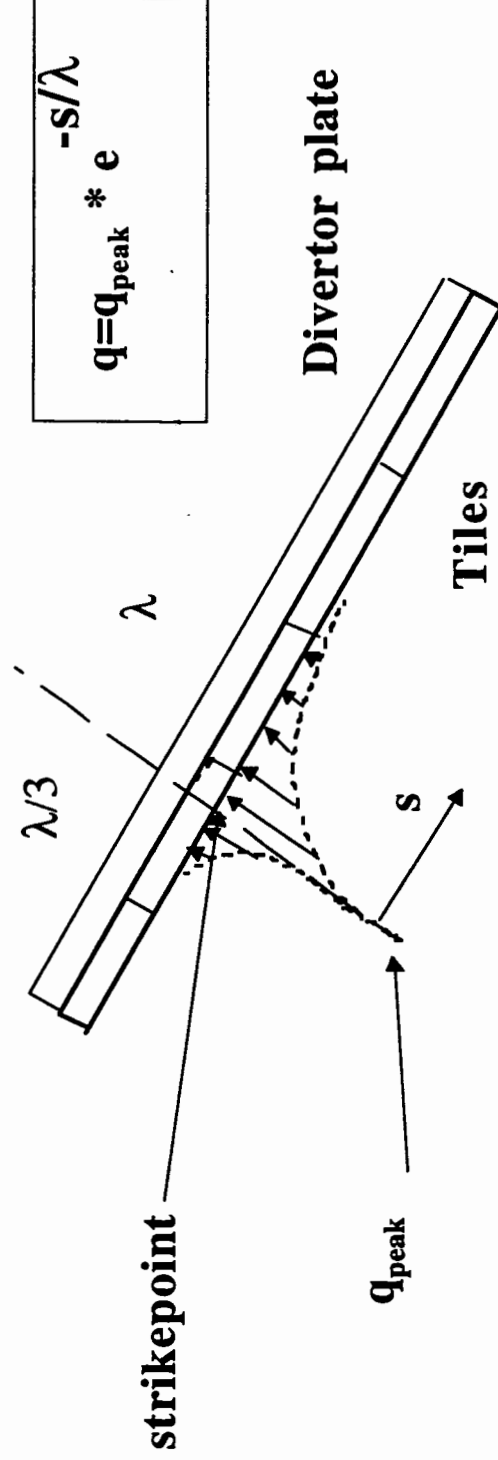
313

Unit = deg C

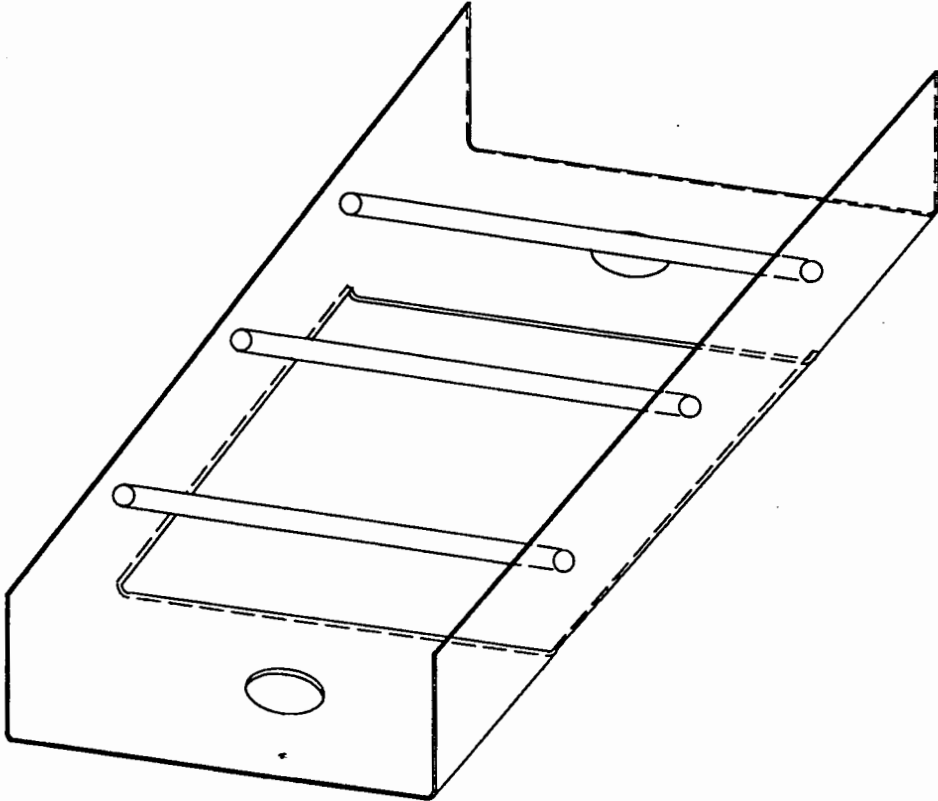
149

Outboard Divertor Plate Cooling/thermal analysis

- Copper plate is cooled at 50 to 100 C and baked out to 350C by Dowtherm
- Analysis shows feasibility for divertor tile operating at
 - ~ 600 W/cm² w/o sweeping
 - ~ 1700 W/cm² with sweeping of 10 cm, where $\lambda = 3$ cm
- Divertor tile can be returned to 150C after pulse in 3 minutes

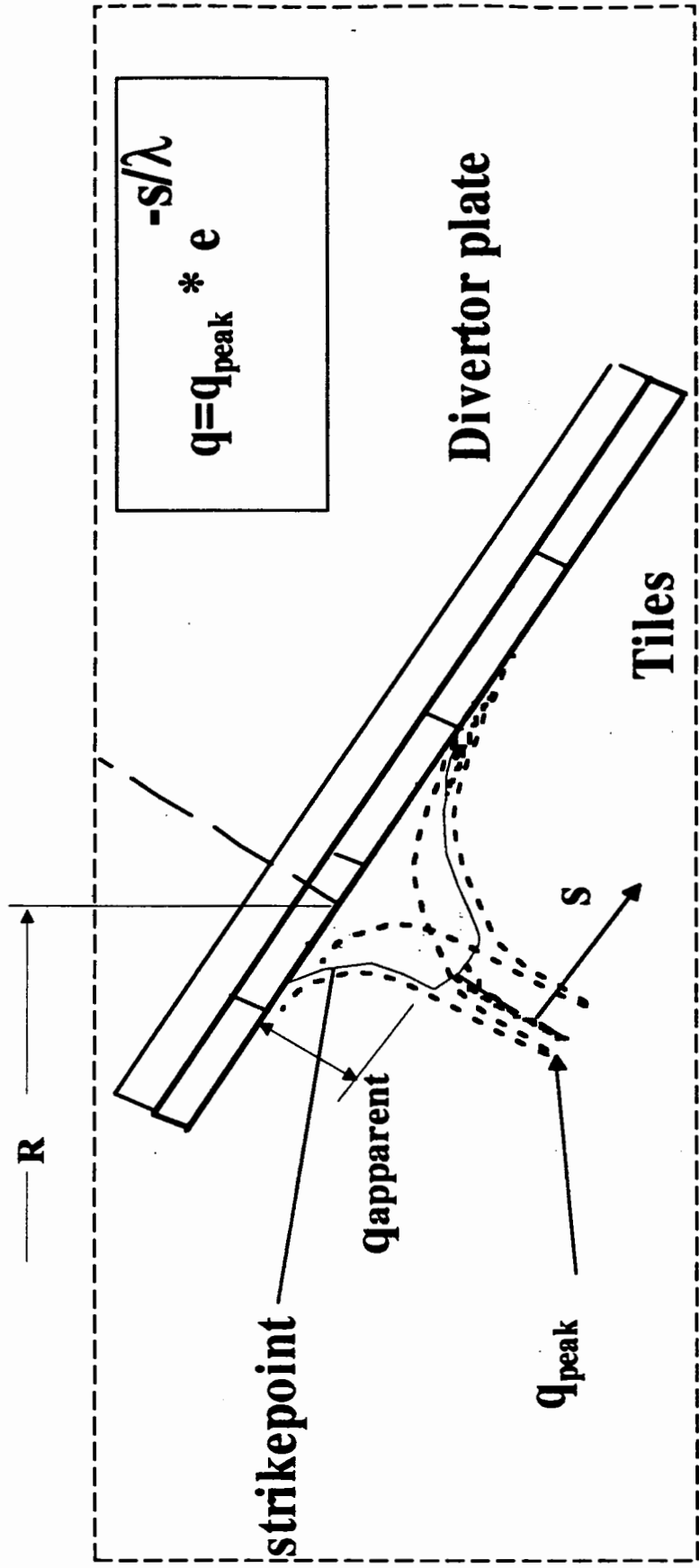


DIVERTOR TILE MOUNTING BRACKET



F. 8

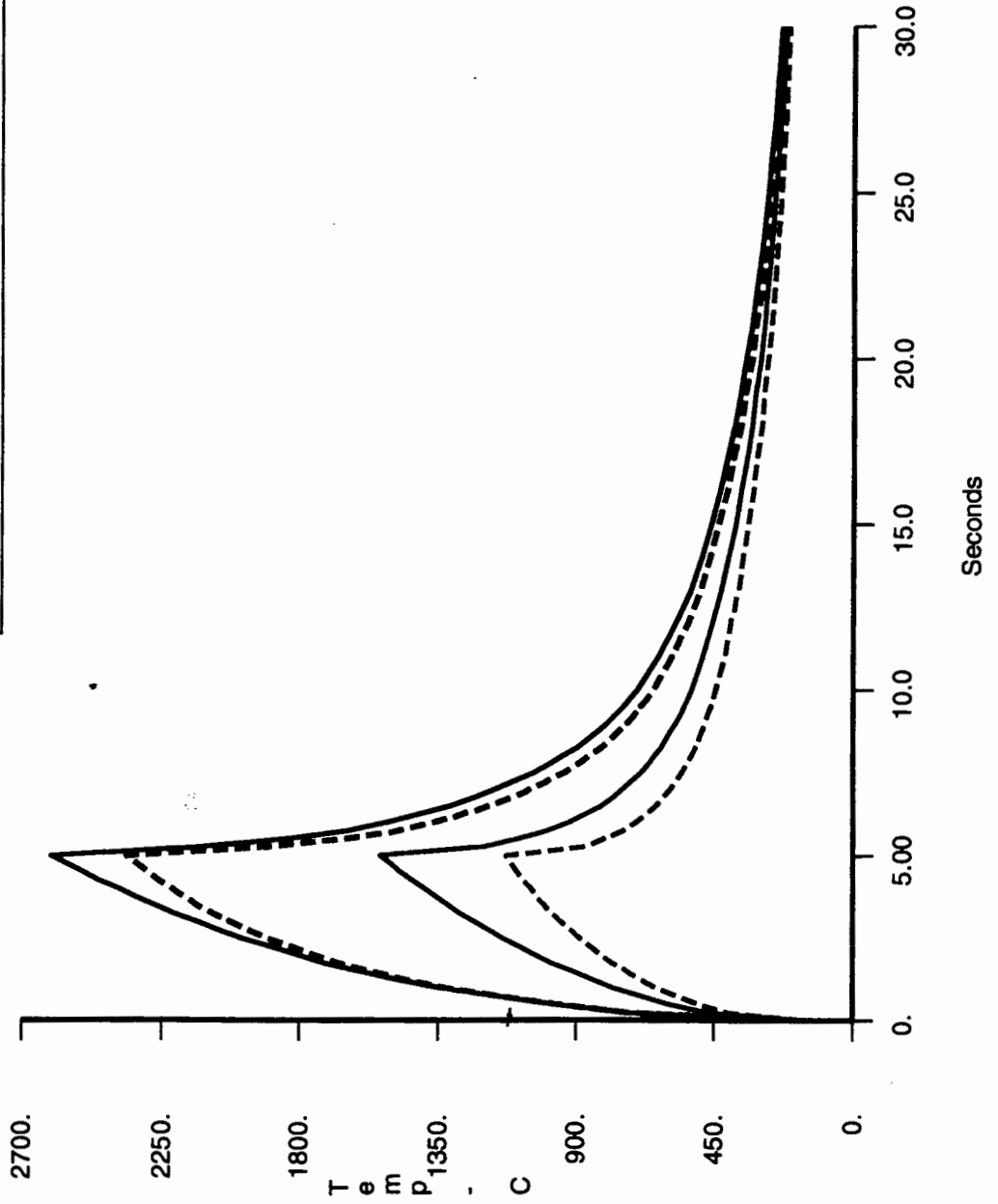
Sweeping will reduce peak flux



NSTX Divertor Tiles

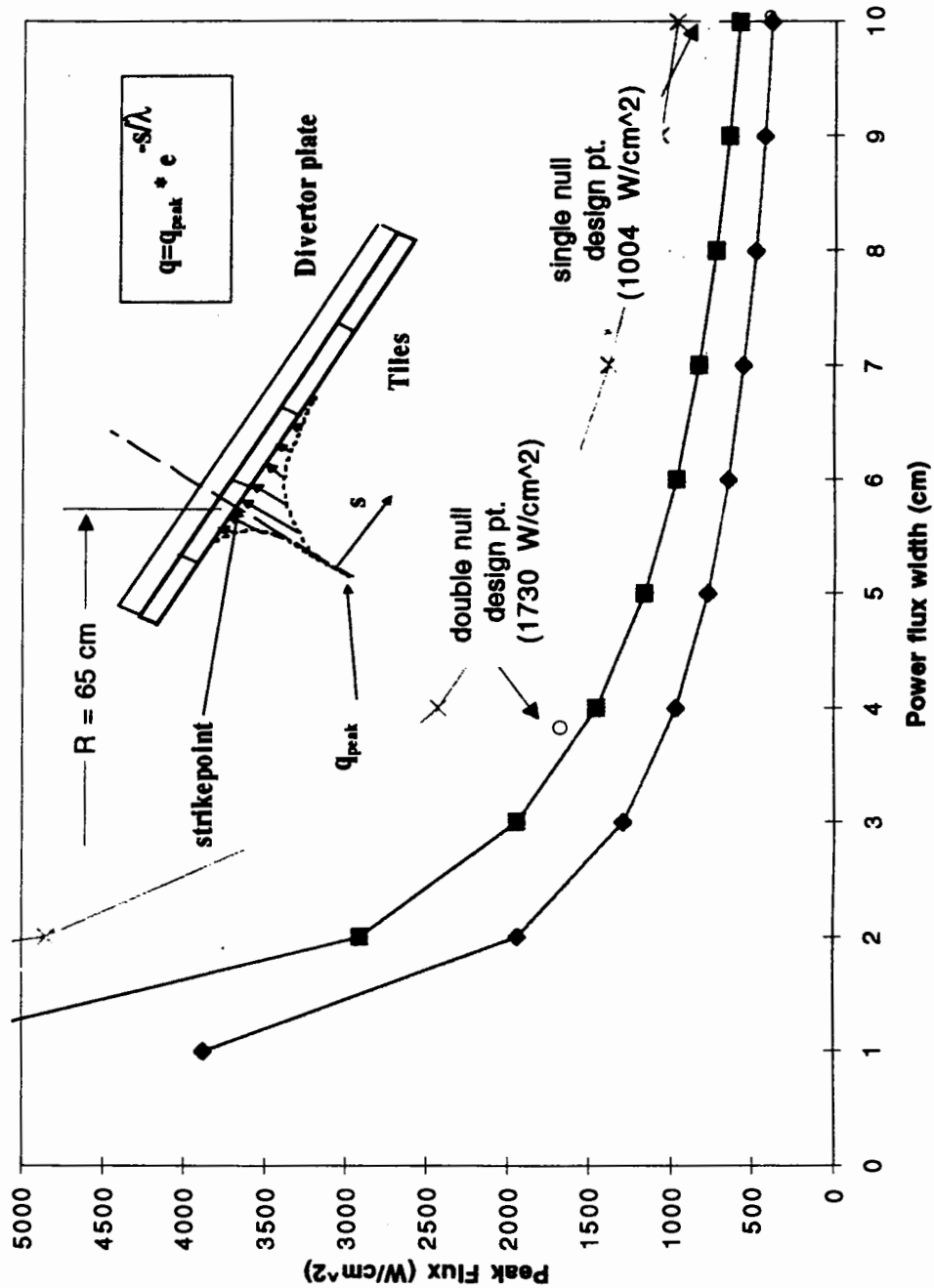
ATJ Graphite
1 inch thickness

LEGEND	
—	Node 1263, Lamda = 1.5 cm
- - - -	Node 1263, Lamda = 1.5 cm, with Radiation
—	Node 1263, Lamda = 3.0
- - - -	Node 1263, Lamda = 4.5



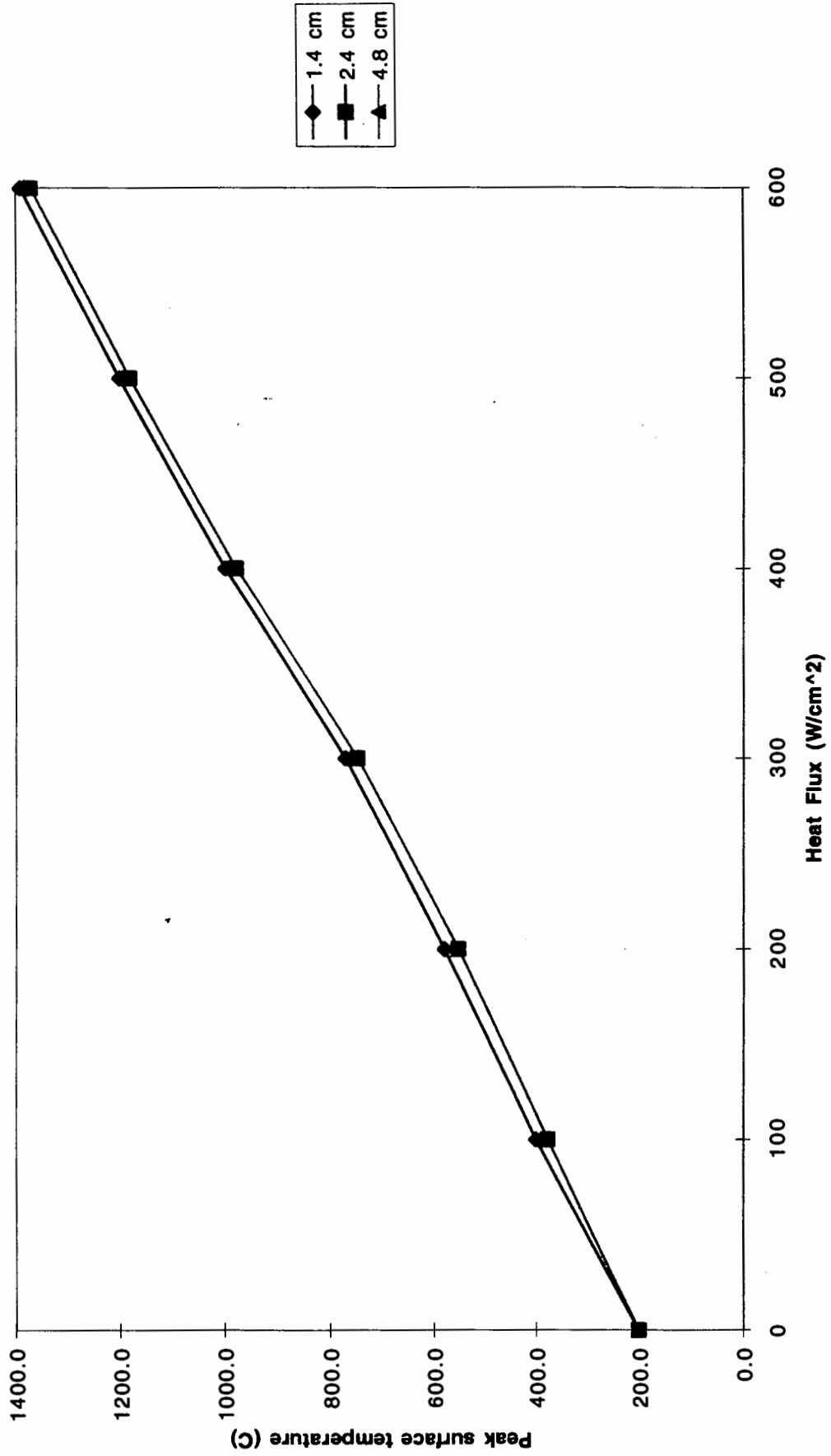
Peak Flux vs Power flux width, one-sided, truncated

(Radius to strike point = 65 cm)



- ◆ 1 MW
- 1.5 MW
- 2 MW
- × 2.5 MW

Peak Surface Temperature vs Heat Flux,
for various tile thicknesses, 1-D calc.
ATJ, 5 seconds on, 300 seconds off,
Fixed equilibrium temperature of 200C



Apparent peak flux vs sweep amplitude, lambda = 3 cm

